



64-bit Intel[®] Xeon[™] Processor MP with 8 MB L3 Cache

Thermal/Mechanical Design Guidelines

March 2005

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1 Introduction

1.1 Objective

The purpose of this guide is to describe the reference thermal solution and design parameters required for the 64-bit Intel® Xeon™ processor MP with up to 8 MB L3 cache. It is also the intent of this document to comprehend and demonstrate the processor cooling solution features and requirements. Furthermore, this document provides an understanding of the processor thermal characteristics, and discusses guidelines for meeting the thermal requirements imposed on the entire life of the processor. The thermal/mechanical solutions described in this document are intended to aid component and system designers to develop and evaluate a processor compatible solution.

1.2 Scope

The thermal/mechanical solutions described in this document pertain only to a solution(s) intended for use with the 64-bit Intel Xeon processor MP with up to 8 MB L3 cache in 2U+ form factor systems. This document contains the mechanical and thermal requirements of the processor (or processor family) cooling solution. In case of conflict, the data in the *64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache Datasheet* supersedes any data in this document. Additional information is provided as a reference in the appendix section(s).

1.3 References

Material and concepts available in the following documents may be beneficial when reading this document.

Document	Comment
<i>64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache Datasheet</i>	http://developer.intel.com
<i>mPGA604 Socket Design Guidelines</i>	http://developer.intel.com
<i>64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache Processor Cooling Solution Mechanical Models</i>	http://developer.intel.com
<i>64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache Mechanical Models</i>	http://developer.intel.com
<i>64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache Thermal Test Vehicle and Cooling Solution Thermal Models</i>	http://developer.intel.com
European Blue Angel Recycling Standards	http://www.blauer-engel.de
Thin Electronics Specification (Server System Infrastructure (SSI) Specification) for Rack Optimized Servers	www.ssiforum.com

NOTE: Contact your Intel field sales representative for the latest revision and order number of this document.

1.4 Definition of Terms

Term	Description
Bypass	Bypass is the area between a passive heatsink and any object that can act to form a duct. For this example, it can be expressed as a dimension away from the outside dimension of the fins to the nearest surface.
FMB	Flexible Motherboard Guideline: an estimate of the maximum value of a processor specification over certain time periods. System designers should meet the FMB values to ensure their systems are compatible with future processor releases.
FSC	Fan Speed Control
IHS	Integrated Heat Spreader: a component of the processor package used to enhance the thermal performance of the package. Component thermal solutions interface with the processor at the IHS surface.
mPGA604	The surface mount Zero Insertion Force (ZIF) socket designed to accept the 64-bit Intel Xeon processor MP with up to 8 MB L3 cache.
P_{MAX}	The maximum power dissipated by a semiconductor component.
Ψ_{CA}	Case-to-ambient thermal characterization parameter (psi). A measure of thermal solution performance using total package power. Defined as $(T_{CASE} - T_{LA}) / \text{Total Package Power}$. Heat source should always be specified for Ψ measurements.
Ψ_{CS}	Case-to-sink thermal characterization parameter. A measure of thermal interface material performance using total package power. Defined as $(T_{CASE} - T_S) / \text{Total Package Power}$.
Ψ_{SA}	Sink-to-ambient thermal characterization parameter. A measure of heatsink thermal performance using total package power. Defined as $(T_S - T_{LA}) / \text{Total Package Power}$.
T_{CASE}	The case temperature of the processor, measured at the geometric center of the topside of the IHS.
T_{CASE_MAX}	The maximum case temperature as specified in a component specification.
TCC	Thermal Control Circuit: Thermal monitor uses the TCC to reduce the die temperature by using clock modulation when the die temperature is very near its operating limits.
$T_{CONTROL}$	A processor unique value, which defines the lower end of the thermal profile and is targeted to be used in fan speed control mechanisms.
Offset	A value programmed into each processor during manufacturing that can be obtained by reading IA_32_TEMPERATURE_TARGET MSR. This is a static and a unique value.
TDP	Thermal Design Power: Thermal solution should be designed to dissipate this target power level. TDP is not the maximum power that the processor/chipset can dissipate.
Thermal Monitor	A feature on the processor that can keep the processor's die temperature within factory specifications under nearly all conditions.
Thermal Profile	Line that defines case temperature specification of a processor at a given power level.
TIM	Thermal Interface Material: The thermally conductive compound between the heatsink and the processor case. This material fills the air gaps and voids, and enhances the transfer of the heat from the processor case to the heatsink.
T_{LA}	The measured ambient temperature locally surrounding the processor. The ambient temperature should be measured just upstream of a passive heatsink or at the fan inlet for an active heatsink.
T_{SA}	The system ambient air temperature external to a system chassis. This temperature is usually measured at the chassis air inlets.
U	A unit of measure used to define server rack spacing height. 1U is equal to 1.75 in, 2U equals 3.50 in, etc.



2 Thermal/Mechanical Reference Design

2.1 Mechanical Requirements

The mechanical performance of the processor cooling solution satisfies the requirements and volumetric keepouts as described in this section.

2.1.1 Performance Target

Table 2-1. Performance Target Table

Parameter	Minimum	Maximum	Unit	Notes
Volumetric Requirements and Keepouts				Refer to drawings in Appendix A
Heatsink Mass		1000 2.2	g lbs	
Static Compressive Load	44 10	222 50	N lbf	1, 2, 3, 4
	44 10	288 65	N lbf	1, 2, 3, 5
Dynamic Compressive Load		222 N + 0.45 kg * 100 G 50 lbf (static) + 1 lbm * 100 G	N lbf	1, 3, 4, 6, 7
		288 N + 0.45 kg * 100 G 65 lbf (static) + 1 lbm * 100 G	N lbf	1,3,5,6,7
Transient		445	N	1,3,8
		100	lbf	

NOTES: In the case of a discrepancy, the most recent processor datasheet supersedes targets listed in the above table.

1. These specifications apply to uniform compressive loading in a direction perpendicular to the IHS top surface.
2. This is the minimum and maximum static force that can be applied by the heatsink and retention solution to maintain the heatsink and processor interface.
3. These parameters are based on limited testing for design characterization. Loading limits are for the package only and do not include the limits of the processor socket.
4. This specification applies for thermal retention solutions that allow baseboard deflection.
5. This specification applies either for thermal retention solutions that prevent baseboard deflection or for the Intel enabled reference solution(COOLING SOLUTION).
6. Dynamic loading is defined as an 11 ms duration average load superimposed on the static load requirement.
7. Experimentally validated test condition used a heatsink mass of 1 lbm (~0.45 kg) with 100 G acceleration measured at heatsink mass. The dynamic portion of this specification in the product application can have flexibility in specific values, but the ultimate product of mass times acceleration should not exceed this validated dynamic load (1 lbm x 100 G = 100 lb).
8. Transient loading is defined as a 2 second duration peak load superimposed on the static load requirement, representative of loads experienced by the package during heatsink installation.

2.1.2 Critical Interface Dimensions (CID)

The CID drawing illustrates the key interfaces between the package and the thermal/mechanical solution for the processor. The drawing is superseded with the drawing in the processor datasheet, should there be any conflicts.

Figure 2-1. Critical Interface Dimensions (Sheet 1 of 2)

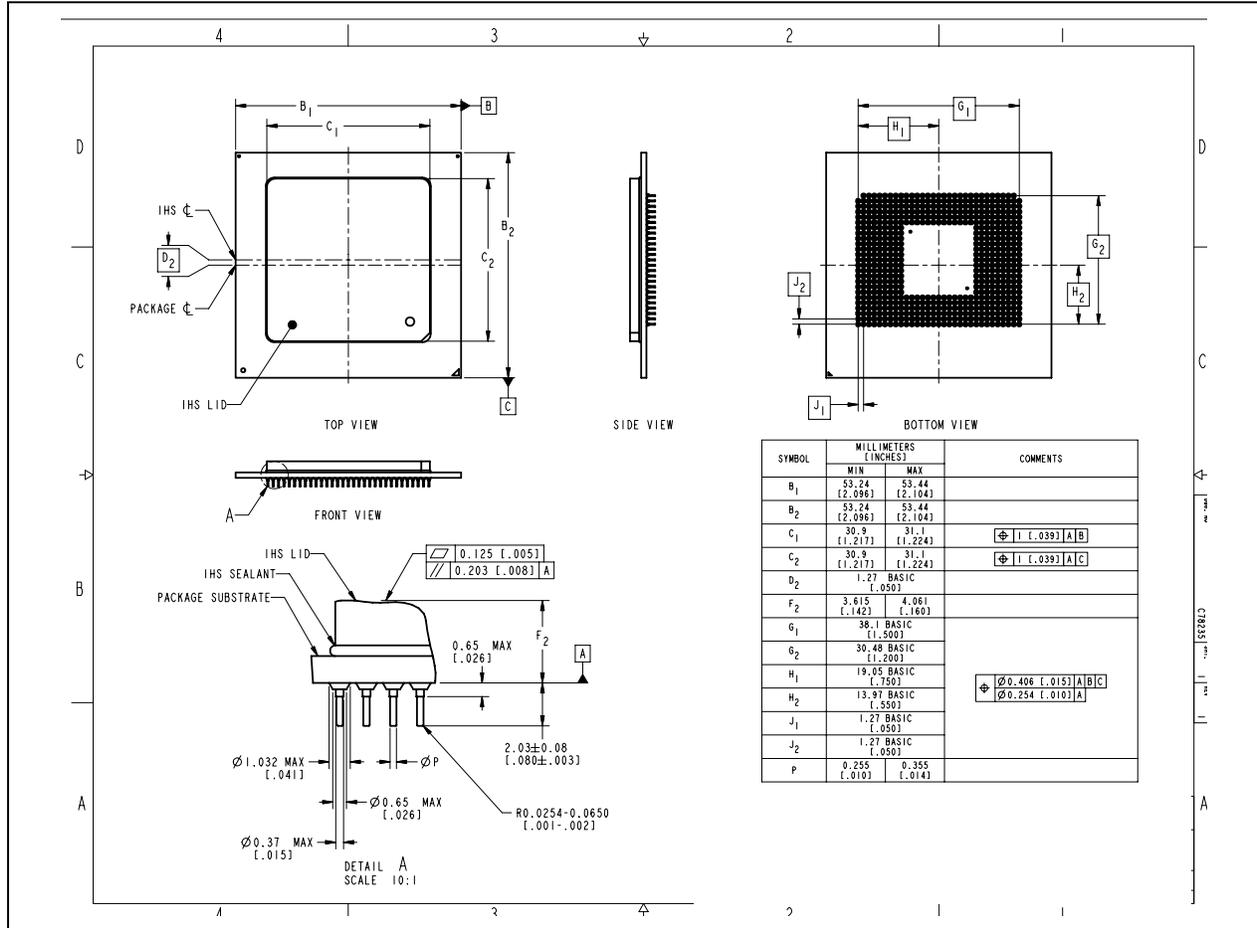
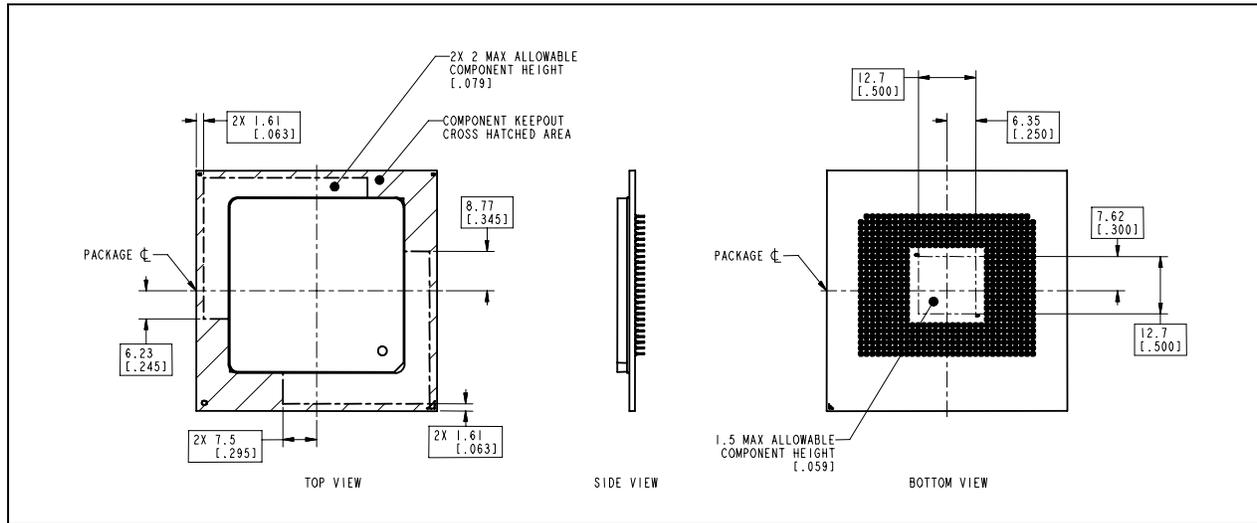


Figure 2-2. Critical Interface Dimensions (Sheet 2 of 2)


2.2 Thermal Requirements

In order to remain within a certain case temperature (T_{CASE}) specification to achieve optimal operation and long-term reliability, the thermal specification methodology, referred to as the thermal profile, is used. The intent of the thermal profile specification is to support acoustic noise reduction through fan speed control and ensure the long-term reliability of the processor.

To ease the burden on thermal solutions, the Thermal Monitor feature and associated logic have been integrated into the silicon of the processor. By taking advantage of the Thermal Monitor feature, system designers may reduce thermal solution cost by designing to Thermal Design Power (TDP) instead of maximum power. The TDP is defined as the power level at which the processor thermal solutions be designed to dissipate. TDP is not the maximum power that the processor/chipset can dissipate. TDP is based on measurements of processor power consumption while running various high power applications. This data set is used to determine those applications that are interesting from a power perspective. These applications are then evaluated in a controlled thermal environment to determine their sensitivity to activation of the thermal control circuit. This data set is then used to derive the TDP targets published in the processor datasheet. Thermal Monitor can protect the processor in rare workload excursions above TDP. Therefore, thermal solutions should be designed to dissipate this target power level.

The relationship between TDP to the thermal profile, and thermal management logic and thermal monitor features, is discussed in the sections to follow. The thermal management logic and thermal monitor features are discussed in extensive detail in the processor Datasheet.

2.2.1 Thermal Profile

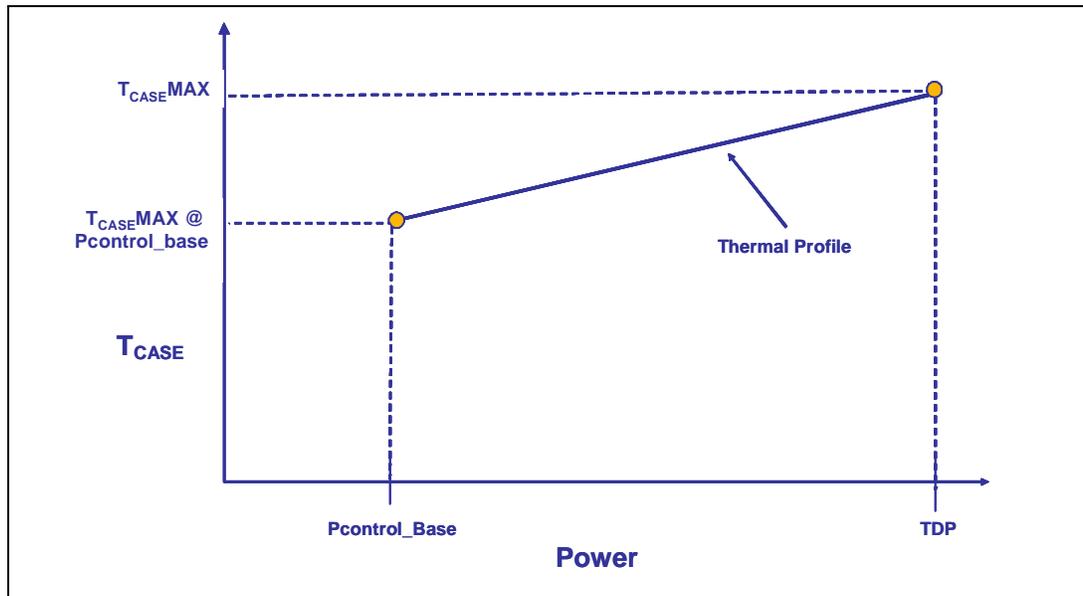
The thermal profile is a linear line that defines the relationship between a processor's case temperature and its power consumption as shown in Figure 2-3. The equation of the thermal profile is defined as:

$$y = ax + b \quad \text{Equation 1}$$

Where:

- y = Processor case temperature, T_{CASE} ($^{\circ}C$)
- x = Processor power consumption (W)
- a = Case-to-ambient thermal resistance, Ψ_{CA} ($^{\circ}C/W$)
- b = Processor local ambient temperature, T_{LA} ($^{\circ}C$)

Figure 2-3. Thermal Profile Diagram



The higher end point of the Thermal Profile represents the processor's TDP and the associated maximum case temperature ($T_{CASE\ MAX}$). The lower end point of the Thermal Profile represents the power value ($P_{control_base}$) and the associated case temperature ($T_{CASE\ MAX\ @\ P_{control_base}}$) for the lowest possible theoretical value of $T_{CONTROL}$ (see Section 2.2.3). This point is also associated with the $T_{CONTROL}$ value defined in Section 2.2.2. The slope of the Thermal Profile line represents the case-to-ambient resistance of the thermal solution with the y-intercept being the local processor ambient temperature. The slope of the Thermal Profile is constant between $P_{CONTROL\ BASE}$ and TDP, which indicate that all frequencies of a processor defined by the Thermal Profile, will require the same heatsink case-to-ambient resistance.

In order to satisfy the Thermal Profile specification, a thermal solution must be at or below the Thermal Profile line for the given processor when its diode temperature is greater than $T_{CONTROL}$ (refer to Section 2.2.2). The Thermal Profile allows the customers to make a trade-off between the thermal solution case-to-ambient resistance and the processor local ambient temperature that best suits their platform implementation (refer to Section 2.3.3). There can be multiple combinations of thermal solution case-to-ambient resistance and processor local ambient temperature that can meet a given Thermal Profile. If the case-to-ambient resistance and the local ambient temperature are known for a specific thermal solution, the Thermal Profile of that solution can easily be plotted against the Thermal Profile specification. As explained above, the case-to-ambient resistance represents the slope of the line and the processor local ambient temperature represents the y-axis intercept. Hence, the T_{CASE} values of a specific solution can be calculated at the TDP and $P_{control_base}$ power levels. Once these points are determined, they can be joined by a line, which represents the Thermal Profile of the specific solution. If that line stays at or below the Thermal Profile specification, then that particular solution is deemed as a compliant solution.

2.2.2 T_{CONTROL} Definition

T_{CONTROL} is a temperature specification based on a temperature reading from the processor's thermal diode. T_{CONTROL} defines the lower end of the Thermal Profile line for a given processor, and it can be described as a trigger point for fan speed control implementation. The value for T_{CONTROL} is calibrated in manufacturing and configured for each processor individually. For the 64-bit Intel Xeon processor MP with up to 8 MB L3 cache the T_{CONTROL} value is obtained by reading a processor model specific register (MSR) and adding this offset value to a base value. The equation for calculating T_{CONTROL} is:

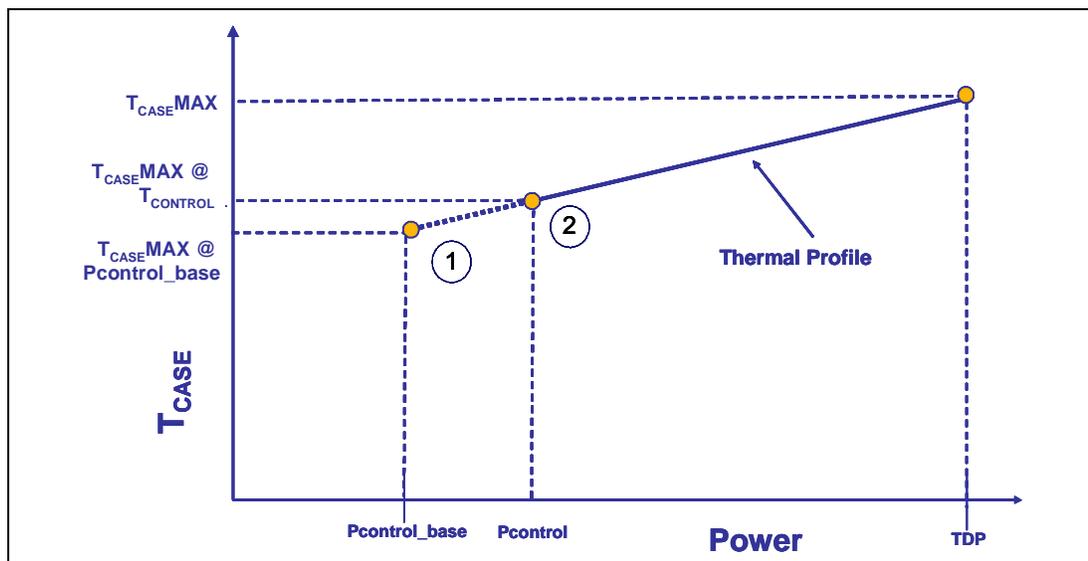
$$T_{\text{CONTROL}} = T_{\text{CONTROL_BASE}} + \text{Offset} \quad \text{Equation 2}$$

Where:

- $T_{\text{CONTROL_BASE}}$ = A fixed base value defined for a given processor generation as published in the processor datasheet.
- Offset = A value programmed into each processor during manufacturing that can be obtained by reading the IA32_TEMPERATURE_TARGET MSR. This is a static and a unique value.

The $T_{\text{CONTROL_BASE}}$ value for this processor is 50°C. The Offset value, which depends on several factors (i.e. leakage current) can be any number between 0 and $(T_{\text{CASE_MAX}} - T_{\text{CONTROL_BASE}})$. Figure 2-4 depicts the interaction between the Thermal Profile and T_{CONTROL} for an Offset value that is greater than 0 (i.e. T_{CONTROL} greater than $T_{\text{CONTROL_BASE}}$).

Figure 2-4. T_{CONTROL} and Thermal Profile Interaction



Since T_{CONTROL} is a processor diode temperature value, an equivalent T_{CASE} temperature must be determined to plot the $T_{\text{CASE_MAX}} @ T_{\text{CONTROL}}$ point on the Thermal Profile graph. Location 1 on the Thermal Profile represents a T_{CASE} value corresponding to an Offset of 0 (the theoretical minimum for the given processor family). Any Offset value greater than 0 moves the point where the Thermal Profile must be met upwards, as shown by location 2 on the graph. If the diode temperature is less than T_{CONTROL} , then the case temperature is permitted to exceed the Thermal Profile, but the diode temperature must remain at or below T_{CONTROL} . In other words, there is no T_{CASE} specification for the processor at power levels less than P_{control} . The thermal solution for the processor must be able to keep the processor's T_{CASE} at or below the T_{CASE} values defined by the

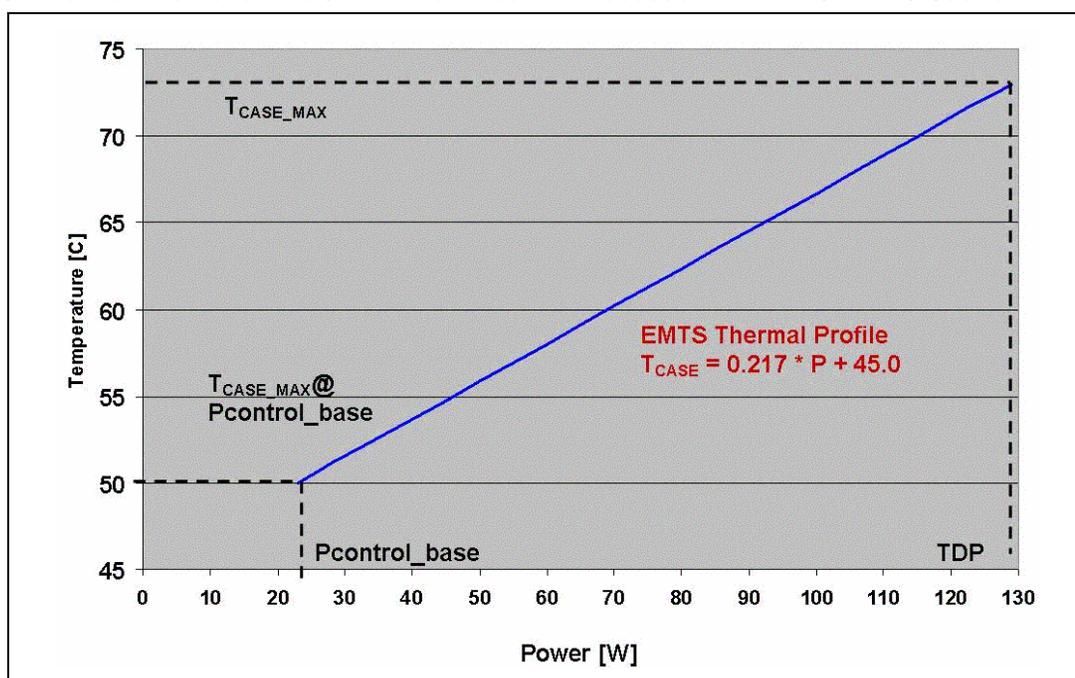
Thermal Profile between the $T_{CASE_MAX}@T_{CONTROL}$ and T_{CASE_MAX} points at the corresponding power levels.

Refer to Section 2.3.1 for the implementation of the $T_{CONTROL}$ value in support of fan speed control (FSC) design to achieve better acoustic performance.

2.2.3 Performance Targets

The Thermal Profile specification for this processor is published in the *64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache Datasheet*. The Thermal Profile specification is shown as a reference in the subsequent discussions.

Figure 2-5. Thermal Profile for the 64-bit Intel® Xeon™ Processor MP with 8 MB L3 Cache



NOTE: The thermal specification shown in this graph is for reference only. Refer to the *64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache Datasheet* for the Thermal Profile specification. In case of conflict, the data information in the datasheet supersedes any data in this figure.

Table 2-2 describes thermal performance targets for the processor cooling solution enabled by Intel.

Table 2-2. Performance Target Table (Sheet 1 of 2)

Parameter	Minimum	Maximum	Unit	Notes
Ψ_{CA}		0.255	°C/W	Mean + 3 sigma
Pressure Drop		0.15	Inches of H2O	
Altitude		Sea-level		Heatsink designed at 0 meters
Airflow		23	CFM	Airflow through the heatsink fins
TIM Dispense Weight		600	mg	Shin-Etsu®G751. Dispense weight is an approximate target.

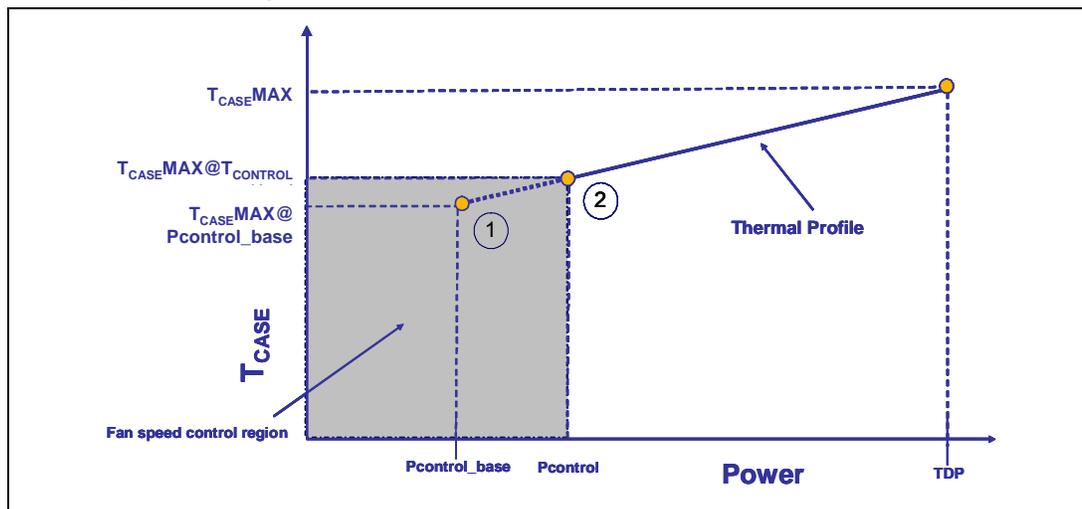
Table 2-2. Performance Target Table (Sheet 2 of 2)

Parameter	Minimum	Maximum	Unit	Notes
TIM Compressive Load	33 147	50 222	lbf N	Generated by the cooling solution.
T_{CASE_MAX}		73	°C	In case of conflict, datasheet supercedes TMDG.
T_{CASE_MAX} @ $P_{control_base}$		50	°C	$P_{control_base} = 23$ W
T_{LA}		40	°C	
TDP		129	W	In case of conflict, datasheet supercedes TMDG.

2.3 Characterizing Cooling Solution Performance Requirements

2.3.1 Fan Speed Control

Fan speed control (FSC) techniques to reduce system level acoustic noise are a common practice in server designs. The fan speed is one of the parameters that determine the amount of airflow provided to the thermal solution. Additionally, airflow is proportional to a thermal solution's performance, which consequently determines the T_{CASE} of the processor at a given power level. Since the T_{CASE} of a processor is an important parameter in the long-term reliability of a processor, the FSC implemented in a system directly correlates to the processor's ability to meet the Thermal Profile and hence the long-term reliability requirements. For this purpose, the parameter called $T_{CONTROL}$ as explained in Section 2.2.2, is to be used in FSC designs to ensure that the long-term reliability of the processor is met while keeping the system level acoustic noise down. Figure 2-6 depicts the relationship between $T_{CONTROL}$ and FSC methodology.

Figure 2-6. $T_{CONTROL}$ and Fan Speed Control


Once the T_{CONTROL} value is determined as explained earlier, the thermal diode temperature reading from the processor can be compared to this T_{CONTROL} value. A fan speed control scheme can be implemented as described in Table 2-3 without compromising the long-term reliability of the processor.

Table 2-3. Fan Speed Control, T_{CONTROL} and T_{DIODE} Relationship

Condition	FSC Scheme
$T_{\text{DIODE}} \leq T_{\text{CONTROL}}$	FSC can adjust fan speed to maintain $T_{\text{DIODE}} \leq T_{\text{CONTROL}}$ (low acoustic region).
$T_{\text{DIODE}} > T_{\text{CONTROL}}$	FSC should adjust fan speed to keep T_{CASE} at or below the Thermal Profile specification (increased acoustic region).

There are many different ways of implementing fan speed control, including FSC based on processor ambient temperature; FSC based on processor thermal diode temperature (T_{DIODE}) or a combination of the two. If FSC is based only on the processor ambient temperature, low acoustic targets can be achieved under low ambient temperature conditions. However, the acoustics cannot be optimized based on the behavior of the processor temperature. If FSC is based only on the thermal diode, sustained temperatures above T_{CONTROL} , drives fans to maximum RPM. If FSC is based both on ambient and thermal diode, ambient temperature can be used to scale the fan RPM controlled by the thermal diode. This would result in an optimal acoustic performance. Regardless of which scheme is employed, system designers must ensure that the Thermal Profile specification is met when the processor diode temperature exceeds the T_{CONTROL} value for a given processor.

2.3.2 Processor Thermal Characterization Parameter Relationships

The idea of a “thermal characterization parameter,” Ψ (psi), is a convenient way to characterize the performance needed for the thermal solution and to compare thermal solutions in identical conditions (heating source, local ambient conditions). A thermal characterization parameter is convenient in that it is calculated using total package power, whereas actual thermal resistance, θ (theta), is calculated using actual power dissipated between two points. Measuring actual power dissipated into the heatsink is difficult, since some of the power is dissipated via heat transfer into the socket and board. Be aware, however, of the limitations of lumped parameters such as Ψ when it comes to a real design. Heat transfer is a three-dimensional phenomenon that can rarely be accurately and easily modeled by lump values.

The case-to-local ambient thermal characterization parameter value (Ψ_{CA}) is used as a measure of the thermal performance of the overall thermal solution that is attached to the processor package. It is defined by the following equation, and measured in units of $^{\circ}\text{C}/\text{W}$:

$$\Psi_{\text{CA}} = (T_{\text{CASE}} - T_{\text{LA}}) / \text{TDP} \quad \text{Equation 3}$$

Where:

- Ψ_{CA} = Case-to-local ambient thermal characterization parameter ($^{\circ}\text{C}/\text{W}$).
- T_{CASE} = Processor case temperature ($^{\circ}\text{C}$).
- T_{LA} = Local ambient temperature in chassis at processor ($^{\circ}\text{C}$).
- P_{D} = TDP dissipation (W) (assumes all power dissipates through the integrated heat spreader (IHS)).

The case-to-local ambient thermal characterization parameter of the processor, Ψ_{CA} , is comprised of Ψ_{CS} , the TIM thermal characterization parameter, and of Ψ_{SA} , the sink-to-local ambient thermal characterization parameter:

$$\Psi_{CA} = \Psi_{CS} + \Psi_{SA} \quad \text{Equation 4}$$

Where:

Ψ_{CS} = Thermal characterization parameter of the TIM ($^{\circ}\text{C}/\text{W}$).

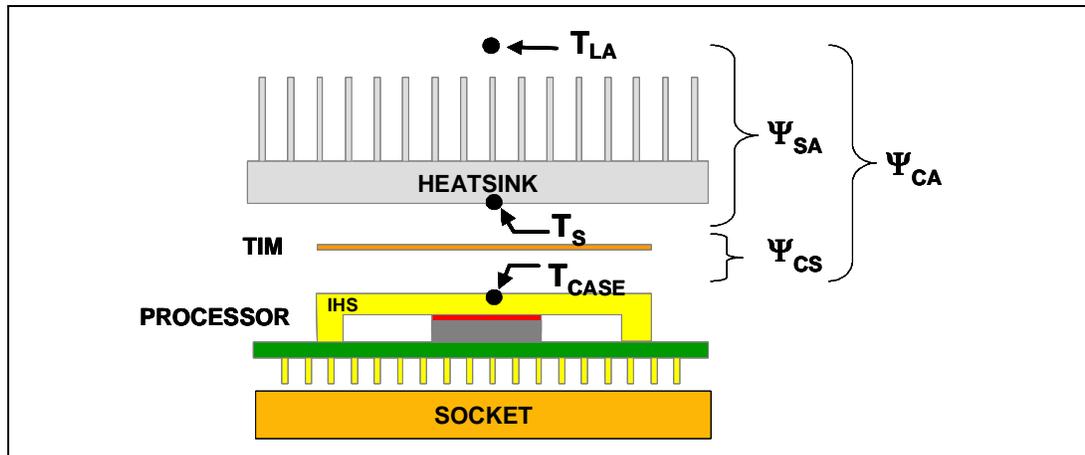
Ψ_{SA} = Thermal characterization parameter from heatsink-to-local ambient ($^{\circ}\text{C}/\text{W}$).

Ψ_{CS} is strongly dependent on the thermal conductivity and thickness of the TIM between the heatsink and IHS.

Ψ_{SA} is a measure of the thermal characterization parameter from the bottom of the heatsink to the local ambient air. Ψ_{SA} is dependent on the heatsink material, thermal conductivity, and geometry. It is also strongly dependent on the air velocity through the fins of the heatsink.

Figure 2-7 illustrates the combination of the different thermal characterization parameters.

Figure 2-7. Processor Thermal Characterization Parameter Relationships



Example

The cooling performance, Ψ_{CA} , is then defined using the principle of thermal characterization parameter described above:

- Define a target case temperature $T_{\text{CASE-MAX}}$ and corresponding TDP at a target frequency, F , given in the processor datasheet.
- Define a target local ambient temperature at the processor, T_{LA} .

Since the processor thermal specifications ($T_{\text{CASE-MAX}}$ and TDP) can vary with the processor frequency, it may be important to identify the worse case (lowest Ψ_{CA}) for a targeted chassis (characterized by T_{LA}) to establish a design strategy such that a given heatsink can cover a given range of processor frequencies.

The following provides an illustration of how one might determine the appropriate performance targets. The example power and temperature numbers used here are not related to any Intel processor thermal specifications, and are for illustrative purposes only.

Assume the datasheet TDP is 85 W and the case temperature specification is 68 °C. Assume as well that the system airflow has been designed such that the local processor ambient temperature is 45 °C. Then the following could be calculated using equation 1 from above for the given frequency:

$$\Psi_{CA} = (T_{CASE} - T_{LA}) / TDP = (68 - 45) / 85 = 0.27 \text{ } ^\circ\text{C/W}$$

To determine the required heatsink performance, a heatsink solution provider would need to determine Ψ_{CS} performance for the selected TIM and mechanical load configuration. If the heatsink solution were designed to work with a TIM material performing at $\Psi_{CS} \leq 0.05 \text{ } ^\circ\text{C/W}$, solving for Equation 2 from above, the performance of the heatsink would be:

$$\Psi_{SA} = \Psi_{CA} - \Psi_{CS} = 0.27 - 0.05 = 0.22 \text{ } ^\circ\text{C/W}$$

If the local processor ambient temperature is assumed to be 40°C, the same calculation can be carried out to determine the new case-to-ambient thermal resistance:

$$\Psi_{CA} = (T_{CASE} - T_{LA}) / TDP = (68 - 40) / 85 = 0.33 \text{ } ^\circ\text{C/W}$$

It is evident from the above calculations that, a reduction in the local processor ambient temperature has a significant positive effect on the case-to-ambient thermal resistance requirement.

2.3.3 Chassis Thermal Design Considerations

2.3.3.1 Chassis Thermal Design Capabilities and Improvements

One of the critical parameters in thermal design is the local ambient temperature assumption of the processor. Keeping the external chassis temperature fixed, internal chassis temperature rise is the only component that can affect the processor local ambient temperature. Every degree gained at the local ambient temperature directly translates into a degree relief in the processor case temperature.

Given the thermal targets for the processor, it is extremely important to optimize the chassis design to minimize the air temperature rise upstream to the processor (T_{RISE}), hence minimizing the processor local ambient temperature.

The heat generated by components within the chassis must be removed to provide an adequate operating environment for both the processor and other system components. Moving air through the chassis brings in air from the external ambient environment and transports the heat generated by the processor and other system components out of the system. The number, size and relative position of fans, vents and other heat generating components determine the chassis thermal performance, and the resulting ambient temperature around the processor. The size and type (passive or active) of the thermal solution and the amount of system airflow can be traded off against each other to meet specific system design constraints. Additional constraints are board layout, spacing, component placement, and structural considerations that limit the thermal solution size.

In addition to passive heatsinks, fan heatsinks and system fans, other solutions exist for cooling integrated circuit devices. For example, ducted blowers, heat pipes and liquid cooling are all capable of dissipating additional heat. Due to their varying attributes, each of these solutions may be appropriate for a particular system implementation.

To develop a reliable, cost-effective thermal solution, thermal characterization and simulation should be carried out at the entire system level, accounting for the thermal requirements of each component. In addition, acoustic noise constraints may limit the size, number, placement, and types of fans that can be used in a particular design.

2.4 Thermal/Mechanical Reference Design Considerations

2.4.1 Heatsink Solutions

2.4.1.1 Heatsink Design Considerations

To remove the heat from the processor, three basic parameters should be considered:

- **The area of the surface on which the heat transfer takes place** - Without any enhancements, this is the surface of the processor package IHS. One method used to improve thermal performance is by attaching a heatsink to the IHS. A heatsink can increase the effective heat transfer surface area by conducting heat out of the IHS and into the surrounding air through fins attached to the heatsink base.
- **The conduction path from the heat source to the heatsink fins** - Providing a direct conduction path from the heat source to the heatsink fins and selecting materials with higher thermal conductivity typically improves heatsink performance. The length, thickness, and conductivity of the conduction path from the heat source to the fins directly impact the thermal performance of the heatsink. In particular, the quality of the contact between the package IHS and the heatsink base has a higher impact on the overall thermal solution performance as processor cooling requirements become strict. Thermal interface material (TIM) is used to fill in the gap between the IHS and the bottom surface of the heatsink, and thereby improves the overall performance of the thermal stackup (IHS-TIM-Heatsink). With extremely poor heatsink interface flatness or roughness, TIM may not adequately fill the gap. The TIM thermal performance depends on its thermal conductivity as well as the pressure load applied to it. Refer to Section 2.4.2 for further information on the TIM between the IHS and the heatsink base.
- **The heat transfer conditions on the surface on which heat transfer takes place** - Convective heat transfer occurs between the airflow and the surface exposed to the flow. It is characterized by the local ambient temperature of the air, T_{LA} , and the local air velocity over the surface. The higher the air velocity over the surface, the resulting cooling is more efficient. The nature of the airflow can also enhance heat transfer via convection. Turbulent flow can provide improvement over laminar flow. In the case of a heatsink, the surface exposed to the flow includes the fin faces and the heatsink base.

An active heatsink typically incorporates a fan that helps manage the airflow through the heatsink.

Passive heatsink solutions require in-depth knowledge of the airflow in the chassis. Typically, passive heatsinks see slower air speed. Therefore these heatsinks are typically larger (and heavier) than active heatsinks due to the increase in fin surface required to meet a required performance. As the heatsink fin density (the number of fins in a given cross-section) increases, the resistance to the airflow increases: it is more likely that the air will travel around the heatsink instead of through it, unless air bypass is carefully managed. Using air-ducting techniques to manage bypass area is an effective method for maximizing airflow through the heat sink fins.

2.4.2 Thermal Interface Material

TIM application between the processor IHS and the heatsink base is generally required to improve thermal conduction from the IHS to the heatsink. Many thermal interface materials can be pre-applied to the heatsink base prior to shipment from the heatsink supplier and allow direct heatsink attach, without the need for a separate TIM dispense or attach process in the final assembly factory.

All thermal interface materials should be sized and positioned on the heatsink base in a way that ensures the entire processor IHS area is covered. It is important to compensate for heatsink-to-processor attach positional alignment when selecting the proper TIM size.

When pre-applied material is used, it is recommended to have a protective application tape over it. This tape must be removed prior to heatsink installation.

The TIM performance is susceptible to degradation (i.e. grease breakdown) during the useful life of the processor due to the temperature cycling phenomena. For this reason, the measured T_{CASE} value of a given processor can decrease over time depending on the type of TIM material.

2.4.3 Summary

In summary, considerations in heatsink design include:

- The local ambient temperature T_{LA} at the heatsink, airflow (CFM), the power being dissipated by the processor, and the corresponding maximum T_{CASE} . These parameters are usually combined in a single lump cooling performance parameter, Ψ_{CA} (case to air thermal characterization parameter). More information on the definition and the use of Ψ_{CA} is given in Section 1.4 and Section 2.3.2.
- Heatsink interface (to IHS) surface characteristics, including flatness and roughness.
- The performance of the TIM used between the heatsink and the IHS.
- Surface area of the heatsink.
- Heatsink material and technology.
- Development of airflow entering and within the heatsink area.
- Physical volumetric constraints placed by the system.

2.4.4 Assembly Overview of the Intel Reference Thermal Mechanical Design

The 2U+ cooling solution reference designed to meet thermal performance targets, and consists of the following components:

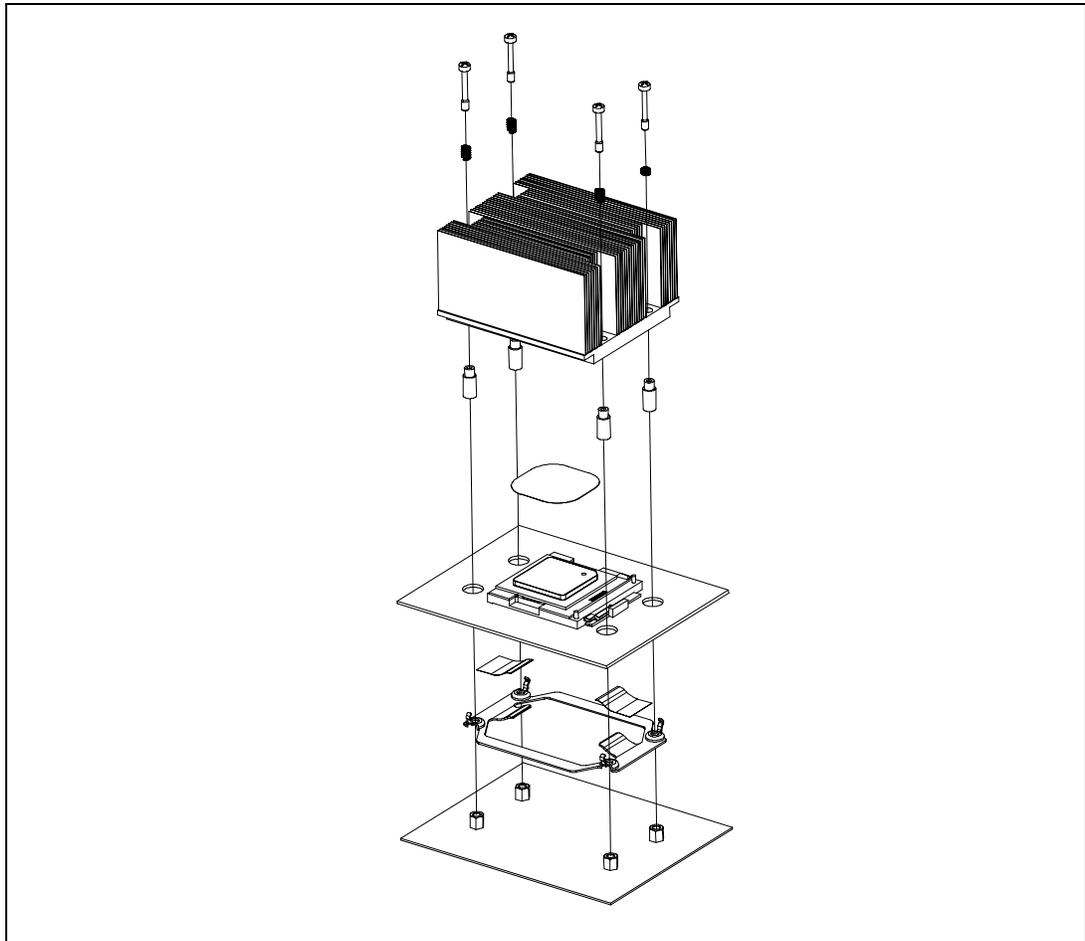
- Heatsink (with captive standoff and screws)
- Thermal Interface Material (TIM)
- Hat Spring

2.4.4.1 Geometric Envelope

The baseboard keepout zones on the primary and secondary sides and height restrictions under the enabling component region are shown in detail in Appendix A. The overall volumetric keep in zone encapsulates the processor, socket, and the entire thermal/mechanical enabling solution.

2.4.4.2 Assembly Drawing

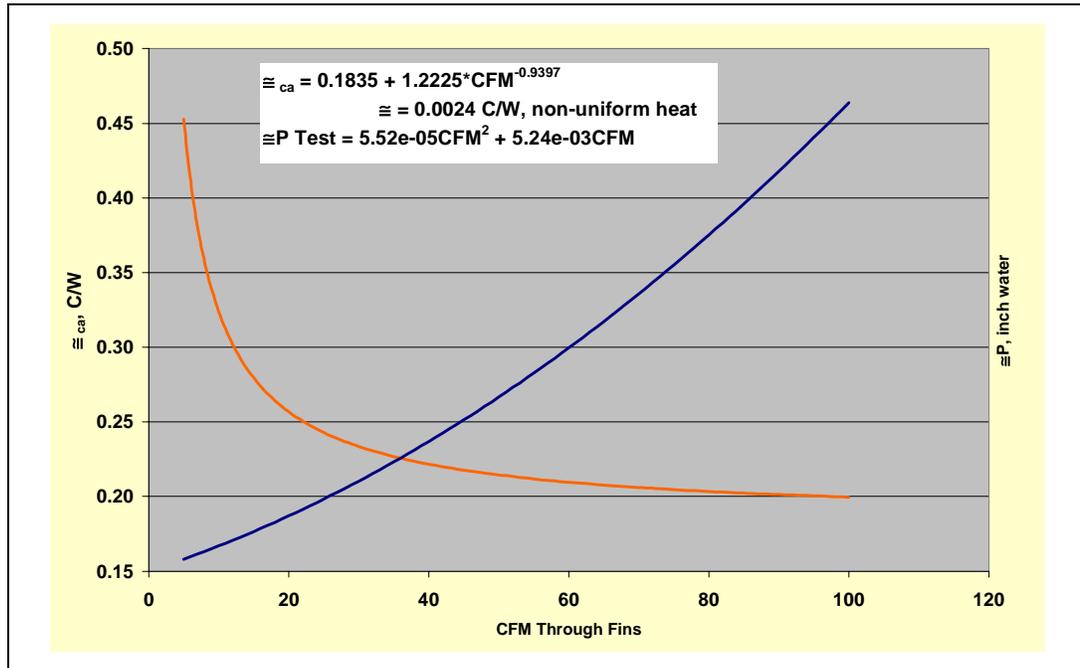
Figure 2-8. Exploded View of Cooling Solution Thermal Solution Components



2.4.5 Thermal Solution Performance Characteristics

The optimization of the cooling solution heatsink for thermal performance is completed and Figure 2-9 shows the thermal performance and the pressure drop through fins of the heatsink versus the airflow provided. The best-fit equations for these curves are also provided to make it easier for users to determine the desired value without any error associated with reading the graph.

Figure 2-9. 2U+ Cooling Solution Heatsink Thermal Performance



If other custom heatsinks are intended for use with the 64-bit Intel Xeon processor MP with up to 8 MB L3 cache, they must support the following interface control requirements to be compatible with the reference mechanical components:

- **Requirement 1:** Heatsink assembly must stay within the volumetric keep-in.
- **Requirement 2:** Maximum mass and center of gravity.

Current maximum heatsink mass is 1000 grams [2.2 lbs] and the maximum center of gravity 3.81 cm [1.5 in.] above the bottom of the heatsink base.

- **Requirement 3:** Maximum and minimum compressive load.

Any custom thermal solution design should meet the loading specification as documented within this document, and should refer to the datasheet for specific details on package loading specifications.

2.4.6 Structural Considerations of Cooling Solution

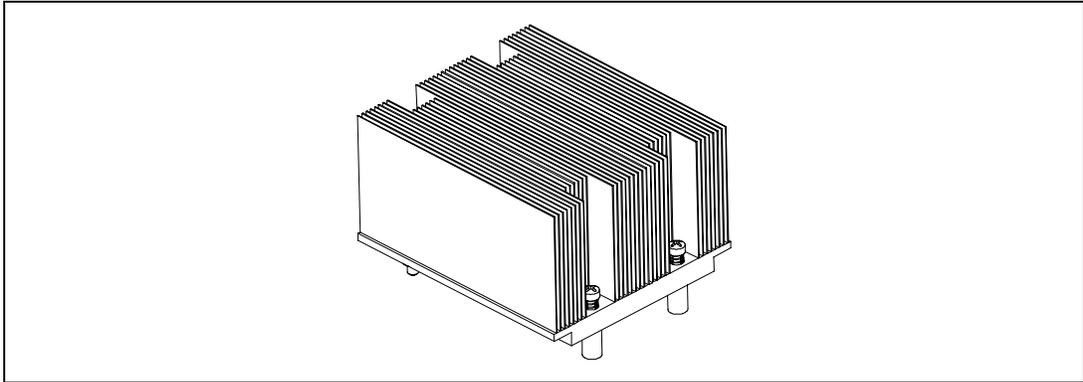
As Intel explores methods of keeping thermal solutions within the air-cooling space, the mass of the thermal solutions is increasing significantly. Due to the flexible nature (and associated large deformation) of baseboard-only attachments, Intel reference solutions, such as cooling solution, are now commonly using direct chassis attach (DCA) as the mechanical retention design. The mass of the new thermal solutions is large enough to require consideration for structural support and stiffening on the chassis. Intel has published a best know method (BKM) document that provides specific structural guidance for designing DCA thermal solutions. The document is titled *Chassis Strength and Stiffness Measurement and Improvement Guidelines for Direct Chassis Attach Solutions*.

2.4.7 Components Overview

2.4.7.1 Heatsink with Captive Screws and Standoffs

The cooling solution reference heatsink uses snapped-fin technology for its design. It consists of a copper base and copper fins with Shin-Etsu* G751 thermal grease as the TIM. The mounting screws and standoffs are also made captive to the heatsink base for ease of handling and assembly as shown in Figure 2-10.

Figure 2-10. Isometric View of the 2U+ Cooling Solution Heatsink



NOTE: Refer to Appendix A for more detailed mechanical drawings of the heatsink.

The function of the standoffs is to provide a bridge between the chassis and the heatsink for attaching and load carrying. When assembled, the heatsink is rigid against the top of the standoff, and the standoff is rigid to a chassis standoff with the hat spring firmly sandwiched between the two. In dynamic loading situations the standoff carries much of the heatsink load, especially in lateral conditions, when compared to the amount of load transmitted to the processor package. As such, it is comprised of steel. The distance from the bottom of the heatsink to the bottom of the standoff is 1.02 cm [0.402 in.].

The function of the screw is to provide a rigid attach method to sandwich the entire cooling solution assembly together, activating the hat spring under the baseboard, and thus providing the TIM preload. A screw is an inexpensive, low profile solution that does not negatively impact the thermal performance of the heatsink due to air blockage. Any fastener (i.e. head configuration) can be used as long as it is of steel construction; the head does not interfere with the heatsink fins, and is of the correct length of 1.27 cm [0.50 in.].

Although the cooling solution heatsink fits into the legacy volumetric keep-in, it has a larger footprint due to the elimination of retention mechanism and clips used in the older enabled thermal/mechanical components. This allows the heatsink to grow its base and fin dimensions, further improving the thermal performance. A drawback of this enlarged size and use of copper for both the base and fins is the increased weight of the heatsink. The cooling solution heatsink is estimated to weigh twice as much as previous heatsinks used with Intel Xeon processors. However, the retention scheme employed by cooling solution is designed to support heavy heatsinks (approximately up to 1000 grams) in cases of shock, vibration and installation as explained in Appendix D.

2.4.7.2 Thermal Interface Material (TIM)

A TIM must be applied between the package and the heatsink to ensure thermal conduction. The cooling solution reference design uses Shin-Etsu* G751 thermal grease.

The recommended grease dispenses weight to ensure full coverage of the processor IHS is given below. For an alternate TIM, full coverage of the entire processor IHS is recommended.

Table 2-4. Recommended Thermal Grease Dispense Weight

Processor	Recommended Thermal Grease	Dispense Weight (mg)
64-bit Intel® Xeon™ Processor MP with up to 8 MB L3 Cache	Shin-Etsu* G751	600

It is recommended that you use thermally conductive grease as the TIM requires special handling and dispense guidelines. The following guidelines apply to Shin-Etsu G751 thermal grease. The use of a semi-automatic dispensing system is recommended for high volume assembly to ensure an accurate amount of grease is dispensed on top of the IHS prior to assembly of the heatsink. A typical dispense system consists of an air pressure and timing controller, a hand held output dispenser, and an actuation foot switch. Thermal grease in cartridge form is required for dispense system compatibility. A precision scale with an accuracy of ± 5 mg is recommended to measure the correct dispense weight and set the corresponding air pressure and duration. The IHS surface should be free of foreign materials prior to grease dispense

Additional recommendations include recalibrating the dispense controller settings after any two hour pause in grease dispense. The grease should be dispensed just prior to heatsink assembly to prevent any degradation in material performance. Finally, the thermal grease should be verified to be within its recommended shelf life before use.

The cooling solution reference solution is designed to apply a compressive load of up to 222 N [50 lbf] on the TIM to improve the thermal performance.

2.4.7.3 Hat Spring

The hat spring, which is attached on the secondary side of the baseboard, is made from 0.80 mm [0.0315 in.] thick 301 stainless steel half hard. Any future versions of the spring will be made from a similar material. The hat spring has four embosses (called “hats”) which, when assembled, rest on the top of the chassis standoffs. The hat spring is located between the chassis standoffs and the heatsink standoffs. The purpose of the hat spring is to provide compressive preload at the TIM interface when the baseboard is pushed down upon it. This spring does not function as a clip of any kind. The two tabs on the spring are used to provide the necessary compressive preload for the TIM when the whole solution is assembled. The tabs make contact on the secondary side of the baseboard. In order to avoid damage to the contact locations on the baseboard, the tabs will be insulated with a 0.127 mm [0.005 in.] thick Kapton* tape (or equivalent). Figure 2-11 shows an isometric view of the hat spring design.

Figure 2-11. Hat Spring Isometric View

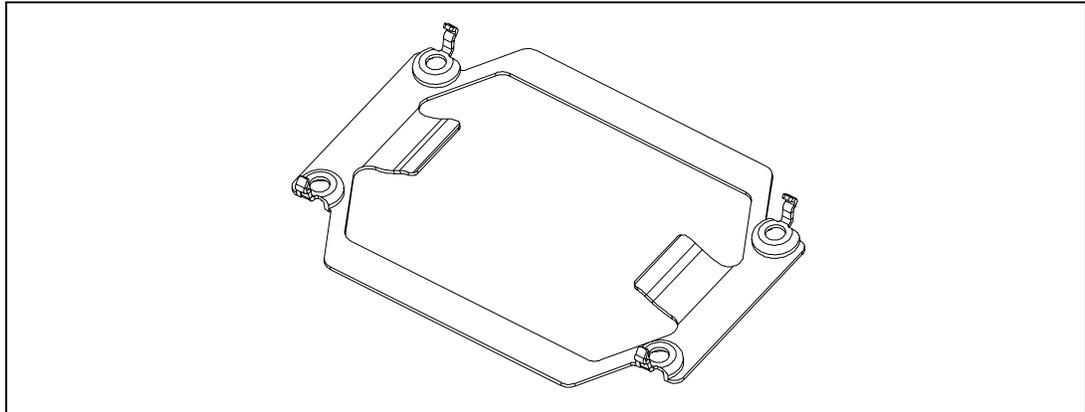
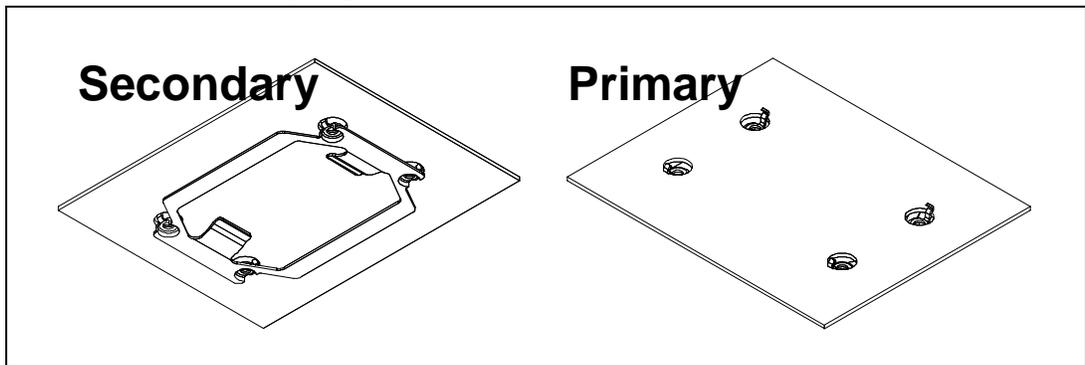


Figure 2-12. Isometric View of Hat Spring Attachment to the Base Board



Please refer to Appendix A for more detailed mechanical drawings of the hat spring. Also, the baseboard keepout requirements shown in Appendix A must be met to use this hat spring design.

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A Mechanical Drawings

The mechanical drawings included in this appendix. These drawings refer to the thermal mechanical enabling components for the 64-bit Intel Xeon processor MP with 8 MB L3 cache.

Note: Intel reserves the right to make changes and modifications to the design as necessary.

Table A-1. Mechanical Drawing List

Drawing Description	Figure Number
2U Cooling Solution Heatsink (Sheet 1 of 4)	Figure A-1
2U Cooling Solution Heatsink (Sheet 2 of 4)	Figure A-2
2U Cooling Solution Heatsink (Sheet 3 of 4)	Figure A-3
2U Cooling Solution Heatsink (Sheet 4 of 4)	Figure A-4
Cooling Solution Hat Spring (Sheet 1 of 3)	Figure A-5
Cooling Solution Hat Spring (Sheet 2 of 3)	Figure A-6
Cooling Solution Hat Spring (Sheet 3 of 3)	Figure A-7
Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 1 of 5)	Figure A-8
Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 2 of 5)	Figure A-9
Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 3 of 5)	Figure A-10
Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 4 of 5)	Figure A-11
Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 5 of 5)	Figure A-12

Figure A-1. 2U Cooling Solution Heatsink (Sheet 1 of 4)

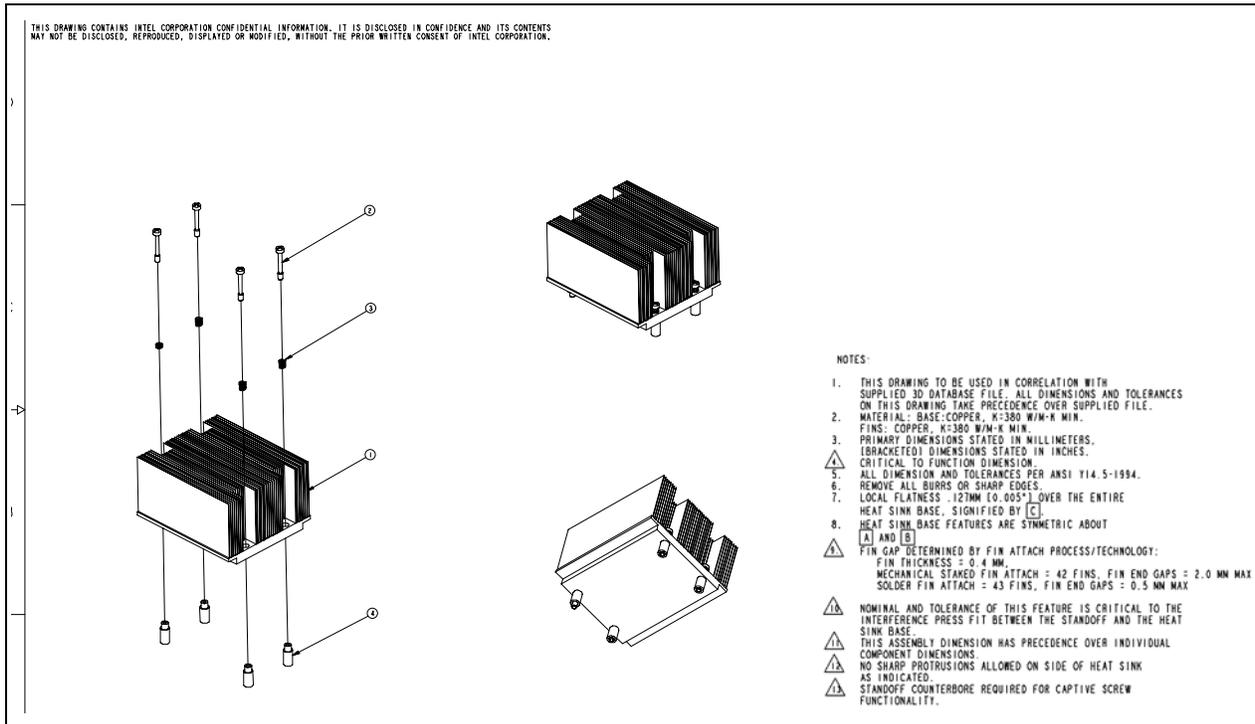


Figure A-2. 2U Cooling Solution Heatsink (Sheet 2 of 4)

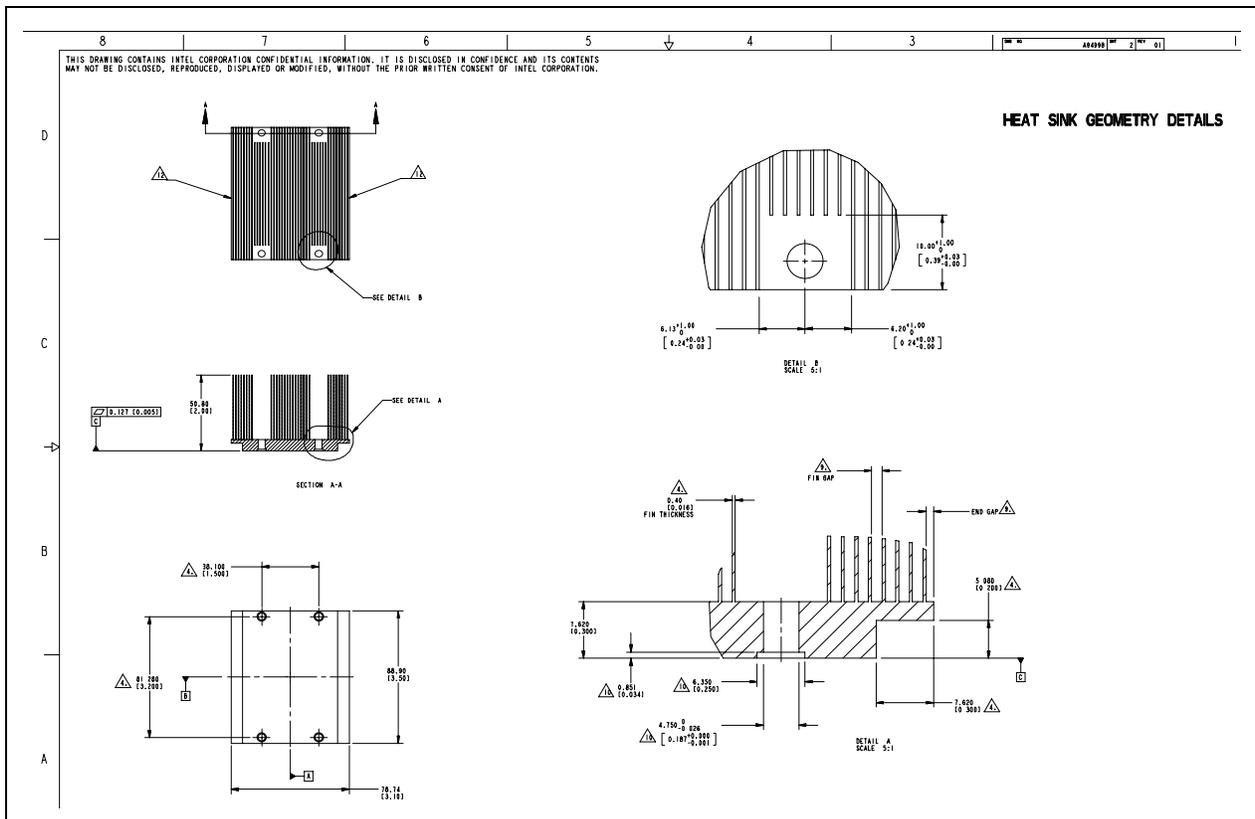


Figure A-3. 2U Cooling Solution Heatsink (Sheet 3 of 4)

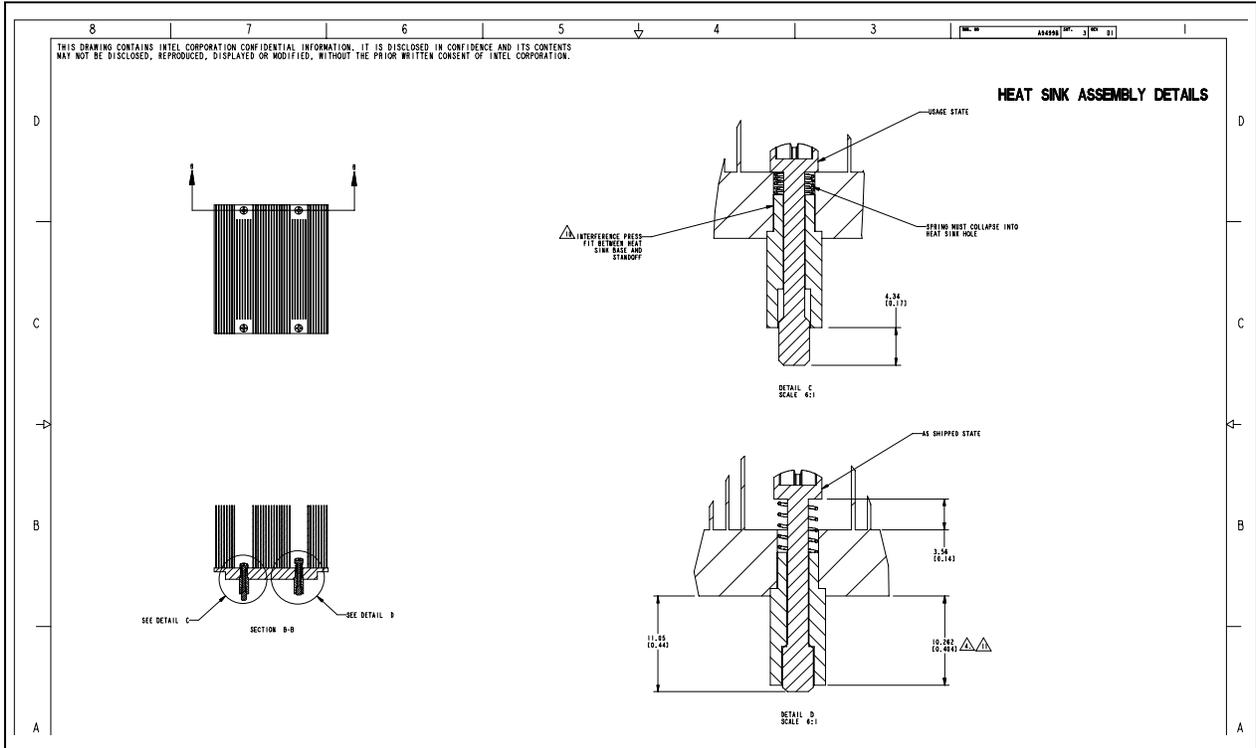


Figure A-4. 2U Cooling Solution Heatsink (Sheet 4 of 4)

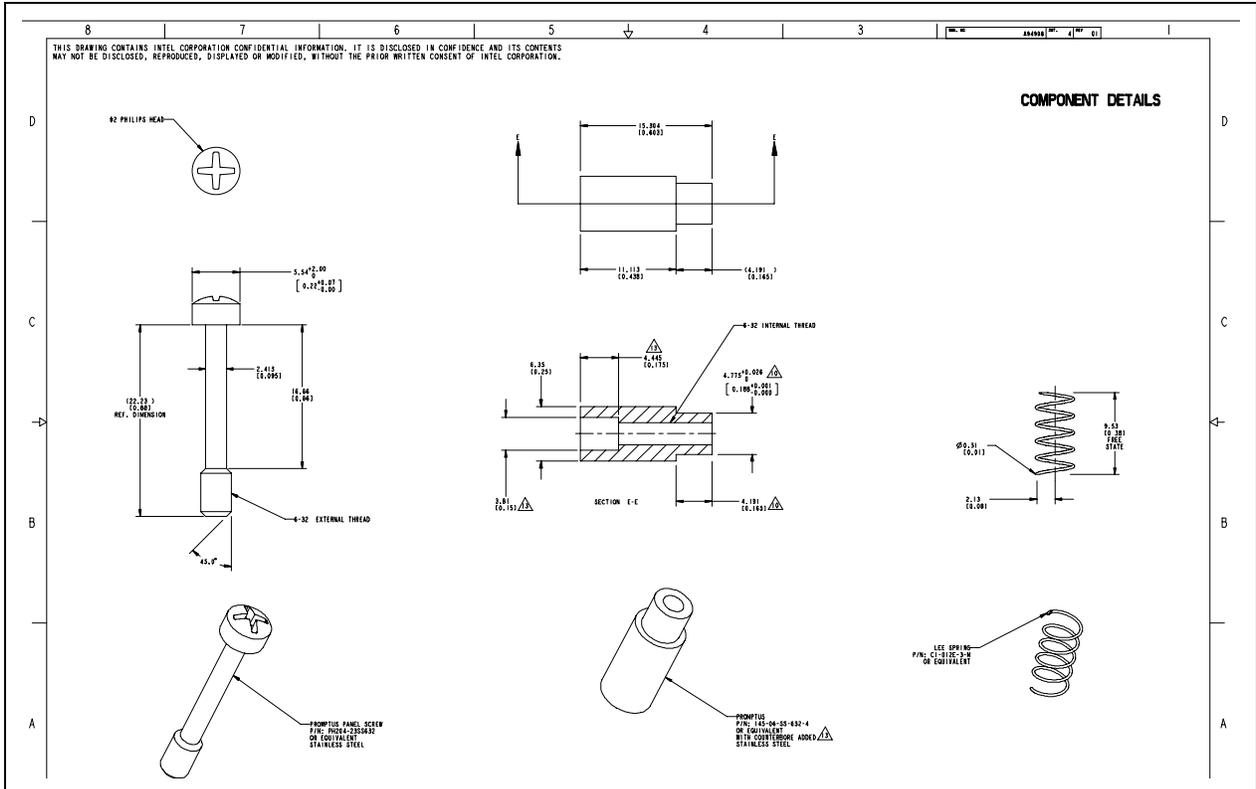


Figure A-7. Cooling Solution Hat Spring (Sheet 3 of 3)

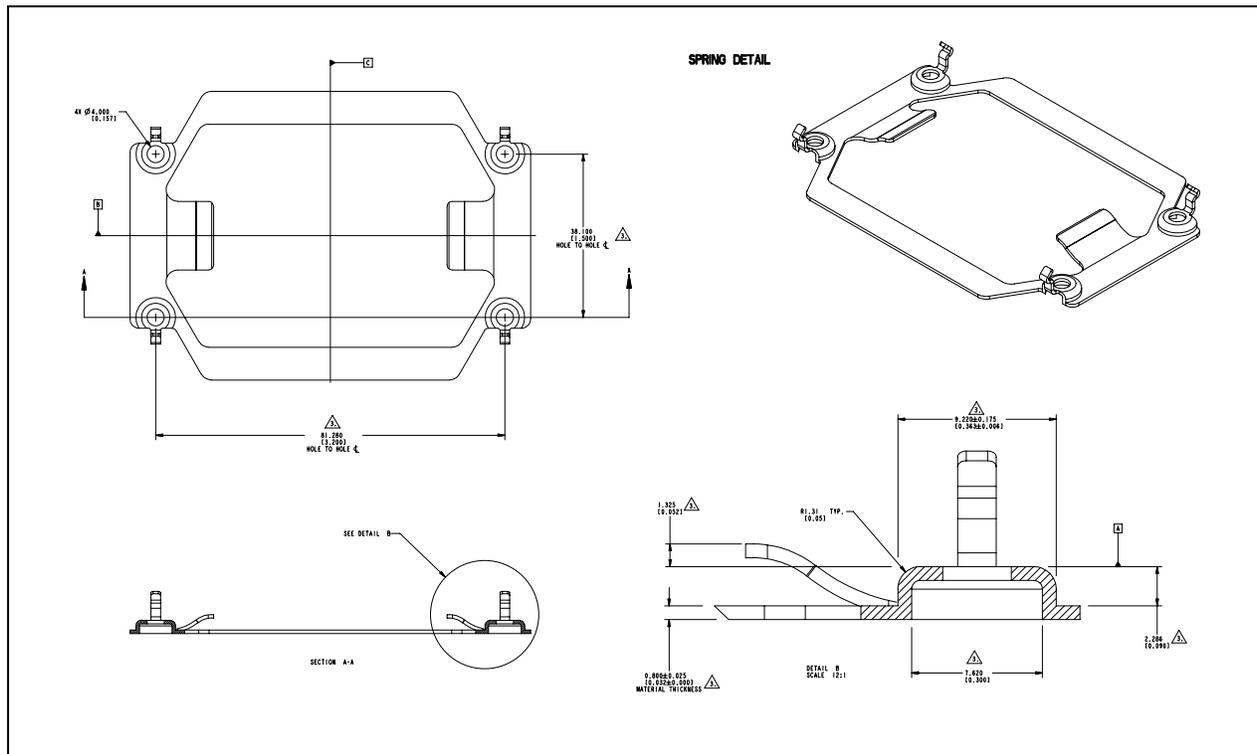


Figure A-8. Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 1 of 5)

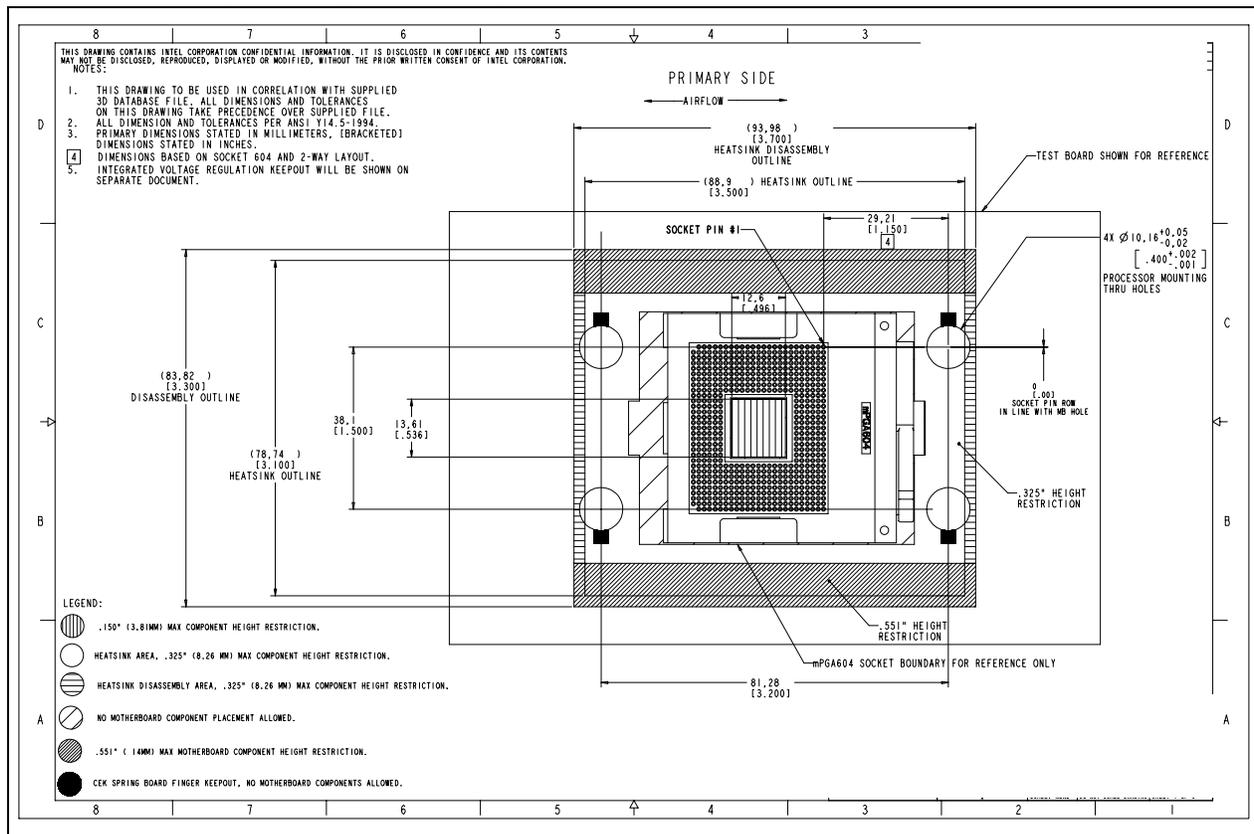


Figure A-10. Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 3 of 5)

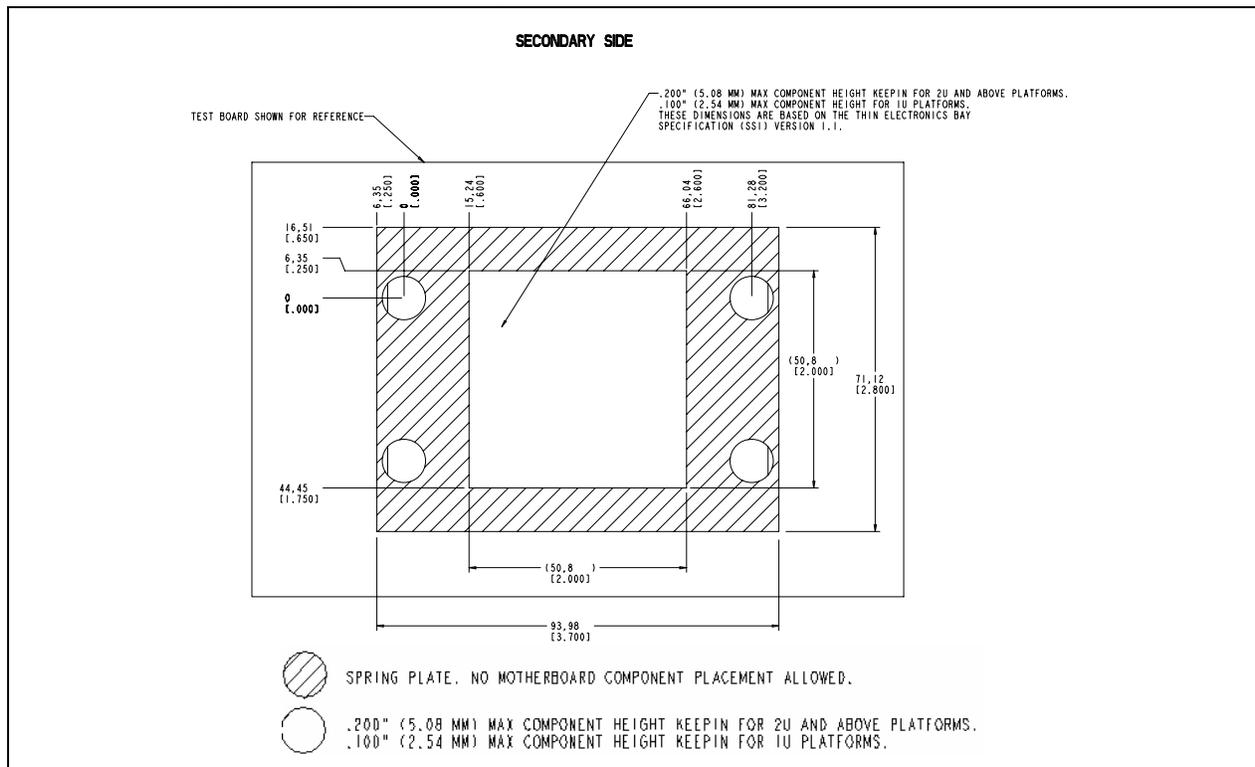


Figure A-11. Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 4 of 5)

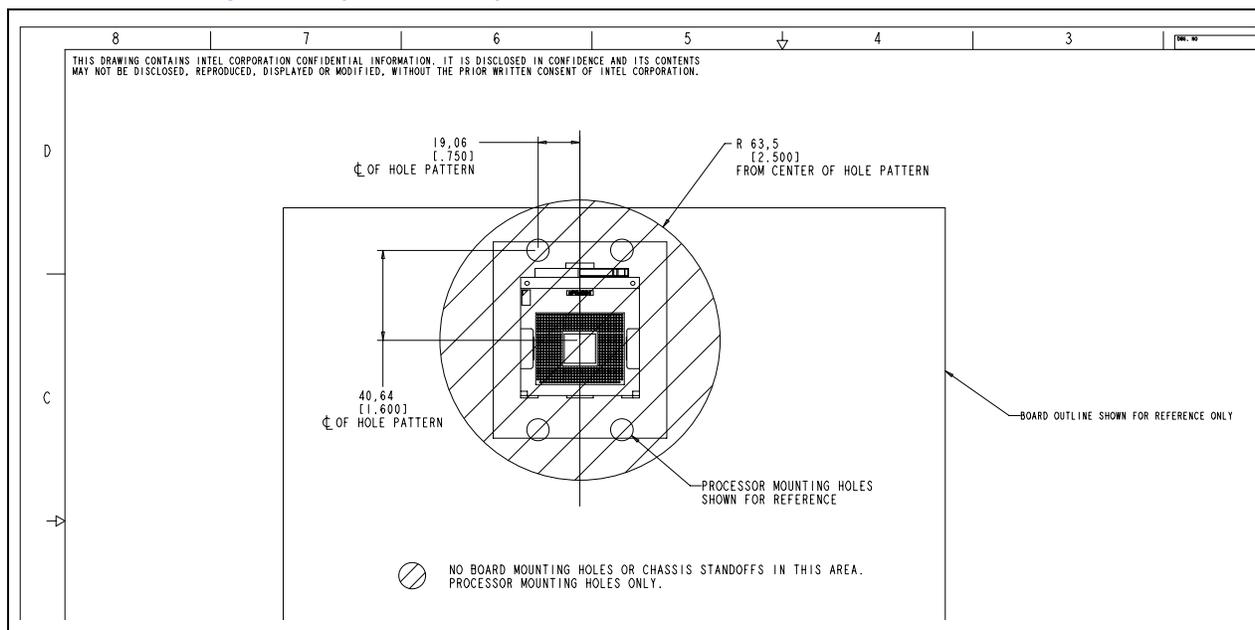
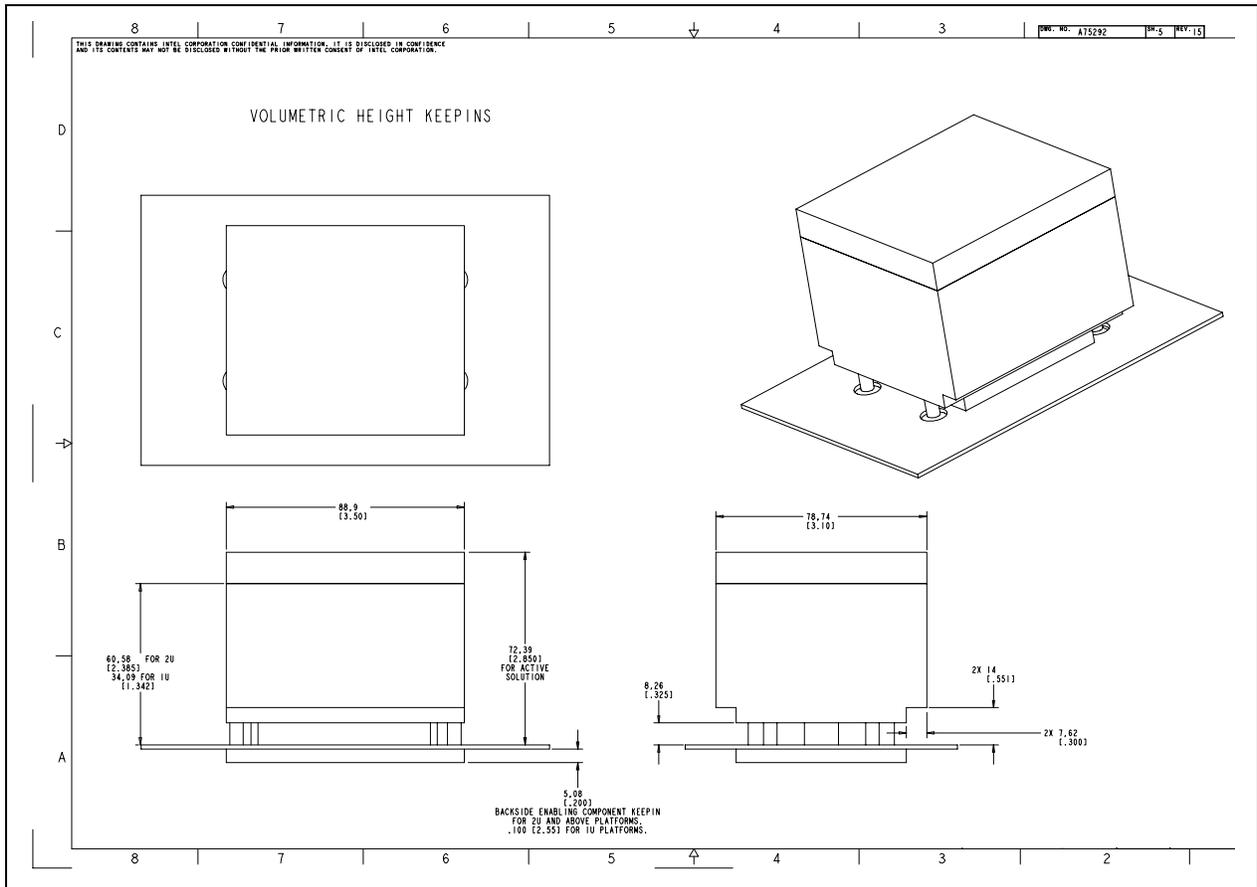


Figure A-12. Baseboard Keepout Footprint Definition and Height Restrictions for Enabling Components (Sheet 5 of 5)



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B Testing Methods

B.1 Case Measurement

Processor cooling performance is determined by measuring the case temperature using a thermocouple. For case temperature measurements, the attach method outlined in this section is recommended for mounting a thermocouple.

Special care is required when measuring case temperature (TC) to ensure an accurate temperature measurement. Thermocouples are often used to measure TC. When measuring the temperature of a surface that is at a different temperature from the surrounding local ambient air, errors could be introduced in the measurements. The measurement errors could be caused by poor thermal contact between the thermocouple junction and the surface of the integrated heat spreader, heat loss by radiation, convection, by conduction through thermocouple leads, or by contact between the thermocouple cement and the heatsink base. To minimize these measurement errors, the approach outlined in the next section is recommended.

B.2 Supporting Test Equipment

To apply the reference thermocouple attach procedure, it is recommended to use the equipment (or equivalent).

Table B-1. Test Equipment

Item	Description	Part Number
Measurement and Output		
Microscope	Olympus* Light microscope or equivalent.	SZ-40
Digital Multi Meter	Digital Multi Meter for resistance measurement.	Not Available
Test Fixture(s)		
Micromanipulator (See Note below)	Micromanipulator set from YOU* Ltd. Or equivalent Mechanical 3D arm with needle (not included) to maintain TC bead location during the attach process.	YOU-3
Miscellaneous Hardware		
Loctite* 498 Adhesive	Super glue w/thermal characteristics.	49850
Adhesive Accelerator	Loctite* 7452 for fast glue curing.	18490
Kapton* Tape	For holding thermocouple in place or equivalent.	Not Available
Thermocouple	Omega*, 36 gauge, "T" Type.	5SRTC-TT-36-72
Calibration and Control		
Ice Point Cell	Omega*, stable 0 C temperature source for calibration and offset.	TRCIII
Hot Point Cell	Omega*, temperature source to control and understand meter slope gain.	CL950-A-110

NOTE: Three axes set consists of (1ea. U-31CF), (1ea. UX-6-6), (1ea. USM6) and (1ea. UPN-1). More information available at: http://www.narishige.co.jp/you_ltd/english/products/set/you-set.htm#3

B.3 Thermal Calibration and Controls

It is recommended that full and routine calibration of temperature measurement equipment be performed before attempting to perform temperature case measurement of processors. Intel recommends checking the meter probe set against known standards. This should be done at 0 °C (using ice bath or other stable temperature source) and at an elevated temperature, around 80 °C (using an appropriate temperature source).

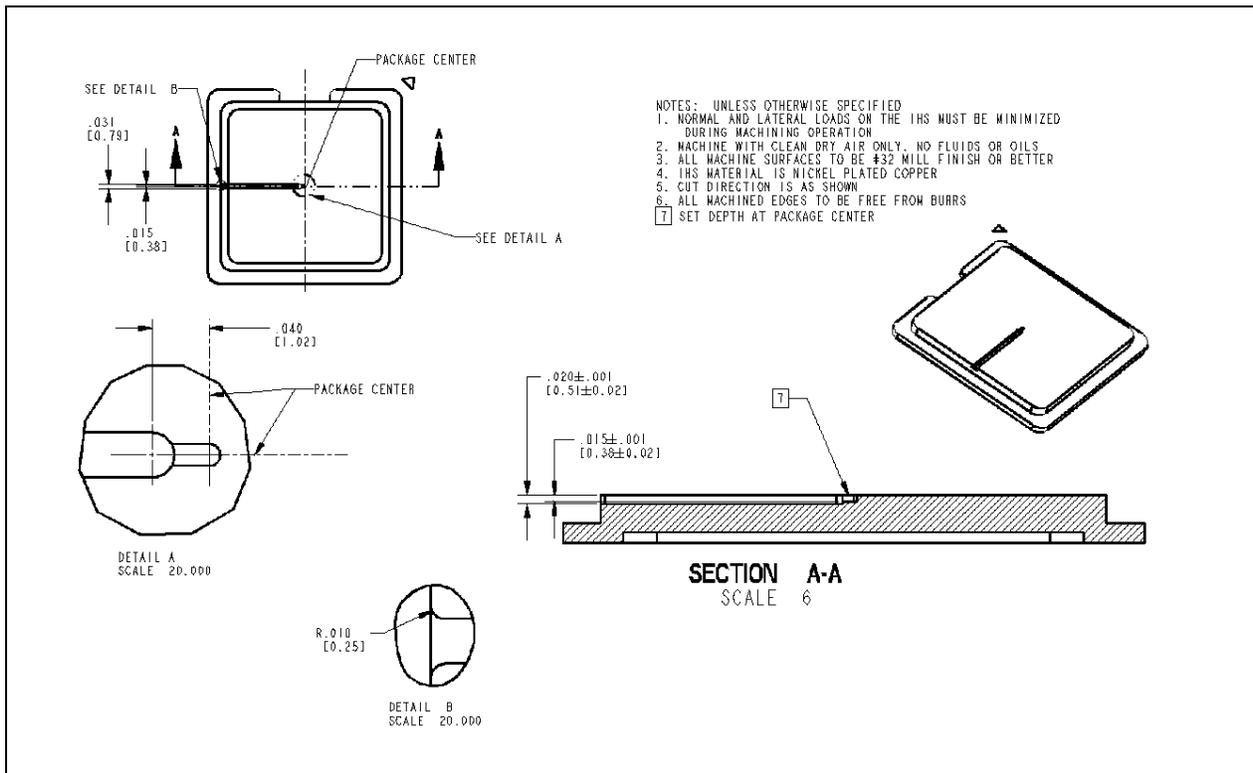
Wire gauge and length also should be considered as some less expensive measurement systems are heavily impacted by impedance. There are numerous resources available throughout the industry to assist with implementation of proper controls for thermal measurements.

1. It is recommended to follow company standard procedures and wear safety items like glasses for cutting the IHS and gloves for chemical handling.
2. Ask your Intel field sales representative if you need assistance to groove and/or install a thermocouple according to the reference process.

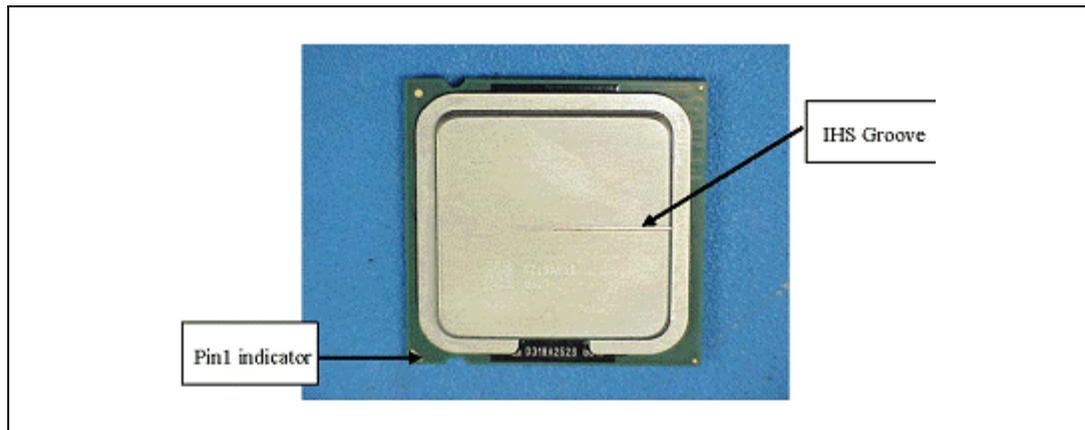
B.4 IHS Groove

Cut a groove in the package IHS according to the drawing.

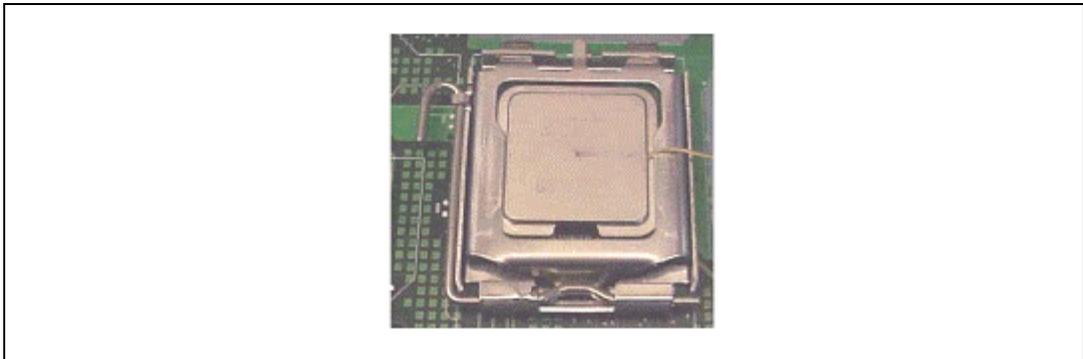
Figure B-1. IHS Groove



NOTE: In case of conflicts in dimensions, the processor datasheet supersedes this information.

Figure B-2. Groove to Pin Indicator

When the processor is installed in the socket, the groove is perpendicular to the socket load lever, and on the opposite side of the lever.

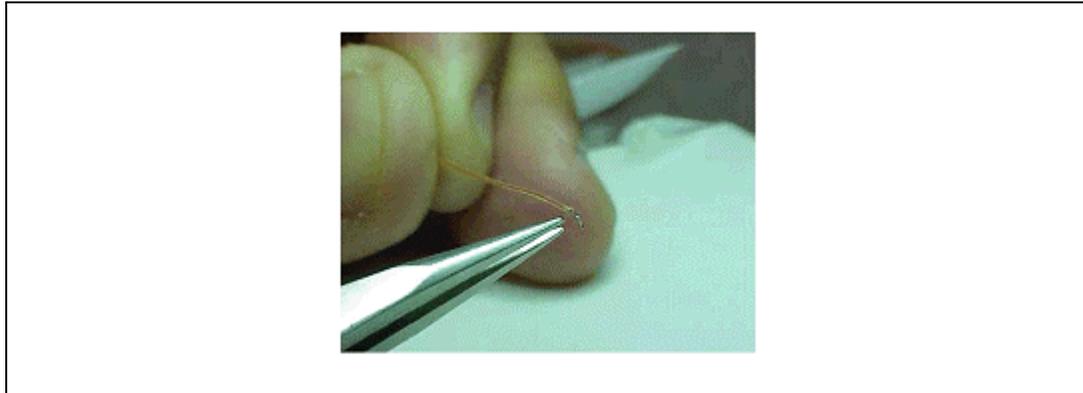
Figure B-3. IHS Groove

Select a machine shop that is capable of holding drawing specified tolerances. IHS channel geometry is critical for repeatable placement of the thermocouple bead, ensuring precise thermal measurements. The specified dimensions minimize the impact of the groove on the IHS under the socket load. A larger groove may cause the IHS to warp under the socket load such that it does not represent the performance of an ungrooved IHS on production packages.

Note: Inspect parts for compliance to specifications before accepting from machine shop.

B.5 Thermocouple Conditioning and Preparation

1. Use a calibrated thermocouple.
2. Measure the thermocouple resistance by holding both wires on one probe and the tip of thermocouple to the other probe of the DMM (compare to thermocouple resistance specifications).
3. Straighten the wire for about 38 mm [1 inch] from the bead to place it inside the channel.
4. Bend the tip of the thermocouple at approximately 45 degree angle by about 0.8 mm [0.030 inch] from the tip.

Figure B-4. Bending Tip of Thermocouple

B.6 Thermocouple Attachment to the IHS

1. Clean groove with IPA and a lint free cloth removing all residues prior to thermocouple attachment.
2. Place the thermocouple wire inside the groove letting the exposed wire and bead extend about 3.2 mm [0.125 inch] past the end of groove. Secure it with Kapton* tape.
3. Lift the wire at the middle of groove with tweezers and bend the front of wire to place the thermocouple in the channel ensuring the tip is in contact with the end of the channel grooved in the IHS.
4. Place the processor under the microscope unit to continue with process. It is also recommended to use a fixture (like processor tray or a plate) to help holding the unit in place for the rest of the attach process.
5. Press the wire down about 6mm [0.125"] from the thermocouple bead using the tweezers. Look in the microscope to perform this task. Place a piece of Kapton* tape to hold the wire inside the groove.
6. Using the micromanipulator, install the needle near to the end of groove on top of thermocouple. Using the X, Y, and Z axes on the arm place the tip of needle on top of the thermocouple bead. Press down until the bead is seated at the end of groove on top of the step.
7. Measure resistance from thermocouple end wires (hold both wires to a DMM probe) to the IHS surface. This should be the same value as measured during the thermocouple conditioning.
8. Place a small amount of Loctite* 498 adhesive in the groove where the bead is installed. Using a fine point device, spread the adhesive in the groove around the needle, the thermocouple bead and the thermocouple wires already installed in the groove during step 5 above. Be careful not to move the thermocouple bead during this step.
9. Measure the resistance from the thermocouple end wires again using the DMM and to ensure the bead is still properly contacting the IHS.

Figure B-5. Securing Thermocouple Wires with Kapton* Tape

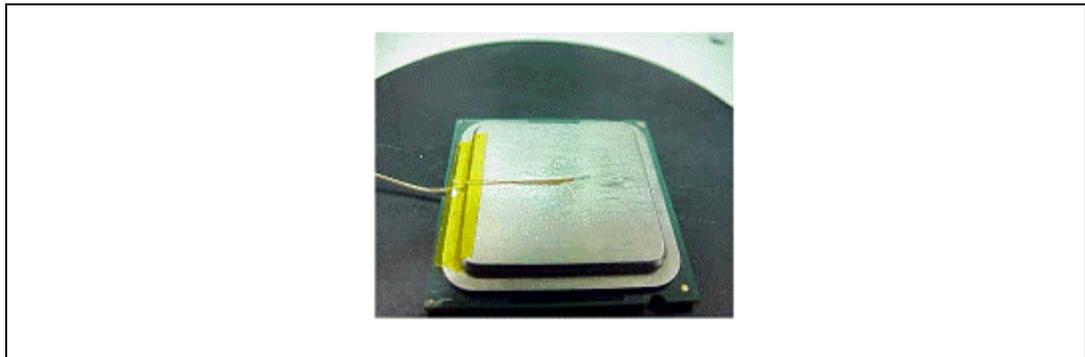


Figure B-6. Thermocouple Bead Placement

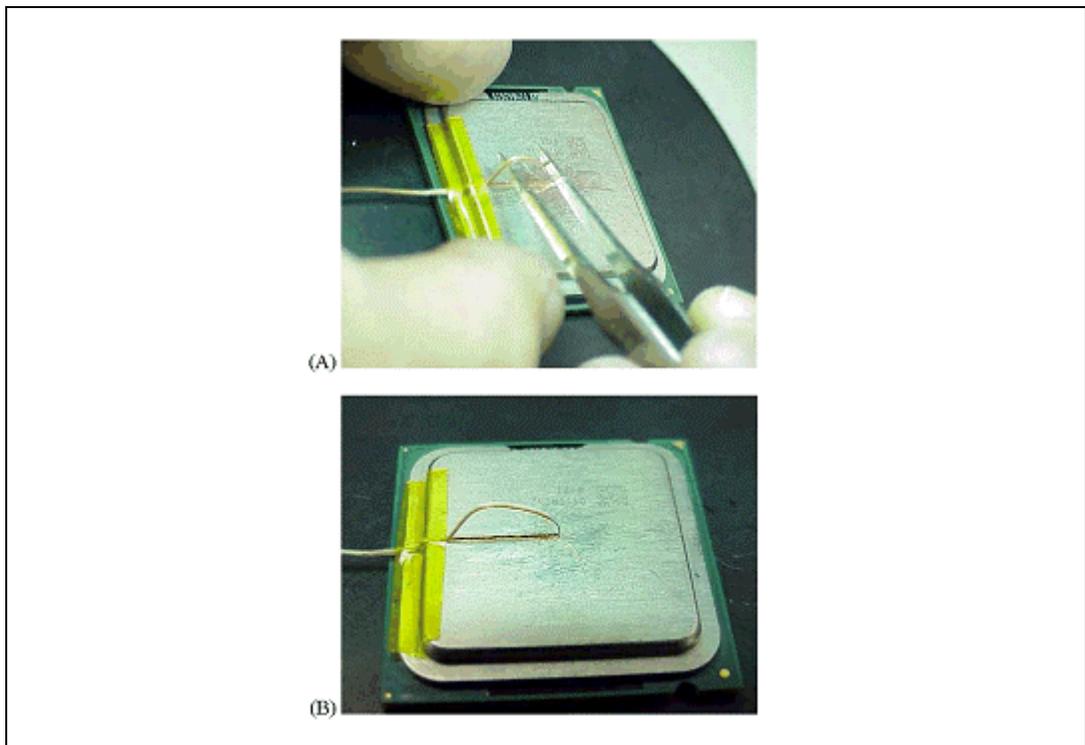


Figure B-7. Thermocouple Placement

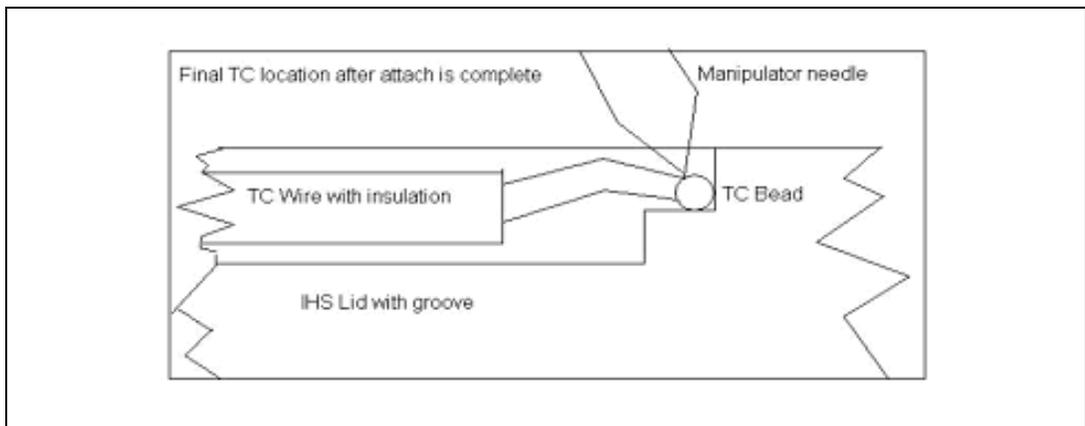


Figure B-8. 3D Micromanipulator to Secure Bead Location



Figure B-9. Measuring Resistance between Thermocouple and IHS

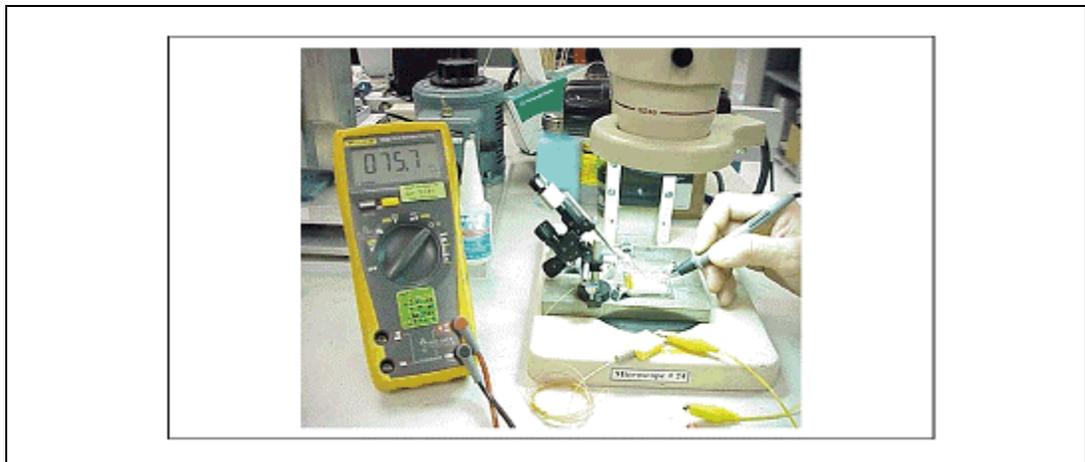
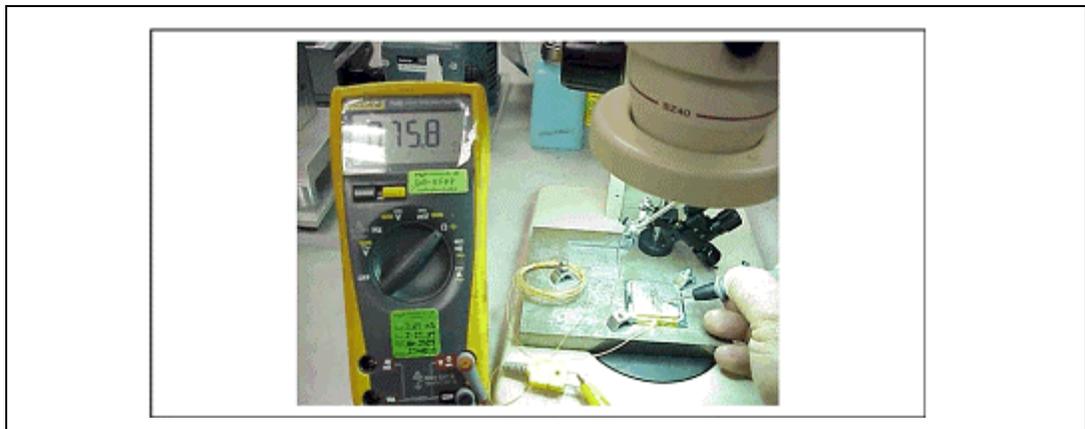


Figure B-10. Applying the Adhesive on the Thermocouple Bead



B.7 Curing Process

1. Let the thermocouple attach set in the open-air for at least 1/2 Hr. It is not recommended to use any curing accelerator like Loctite* Accelerator 7452 for this step, as rapid contraction of the adhesive during curing may weaken bead attach on the IHS.
2. Reconfirm electrical connectivity with DMM before removing the micromanipulator.
3. Remove the 3D Arm needle by holding down the processor unit and lifting the arm.
4. Remove the Kapton* tape, straighten the wire in the groove so it lays flat all the way to the end of the groove.
5. Using a blade to shave excess adhesive above the IHS surface.

Take usual precautions when using open blades.

1. Install new Kapton* tape to hold the thermocouple wire down and fill the rest of groove with adhesive. Make sure the wire and insulation is entirely within the groove and below the IHS surface.
2. Curing time for the rest of the adhesive in the groove can be reduced using Loctite* Accelerator 7452.
3. Repeat step 5 to remove any access adhesive to ensure flat IHS for proper mechanical contact to the heatsink surface.

B.8 Thermocouple Wire Management

Figure B-11. Thermocouple Wire Management in the Groove

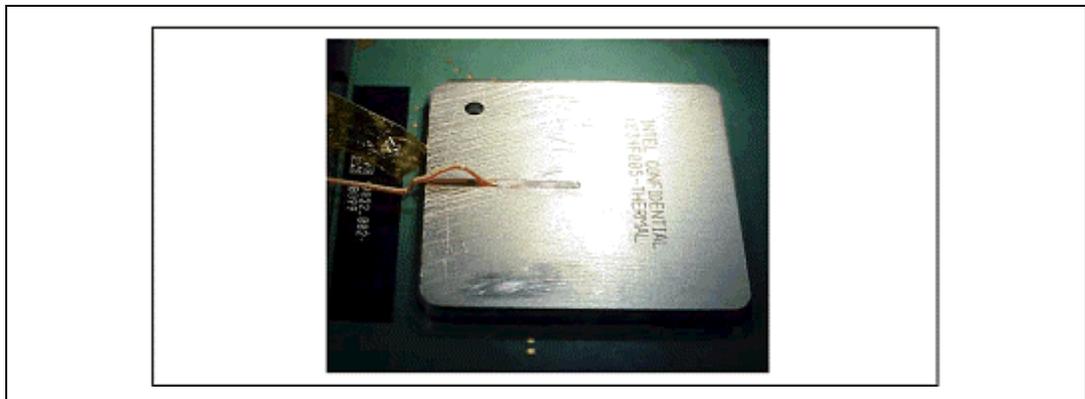
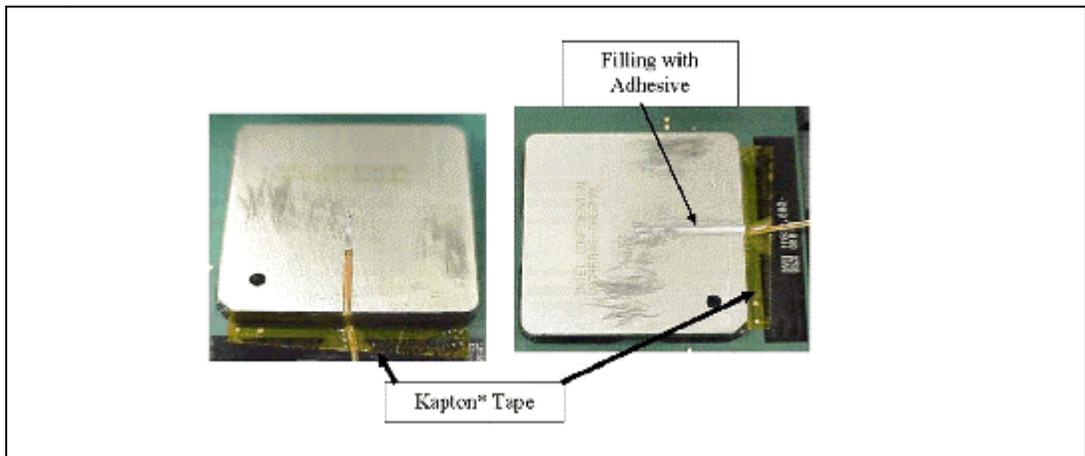


Figure B-12. Removing Excess Adhesive from the IHS



Figure B-13. Filling the Groove with Adhesive

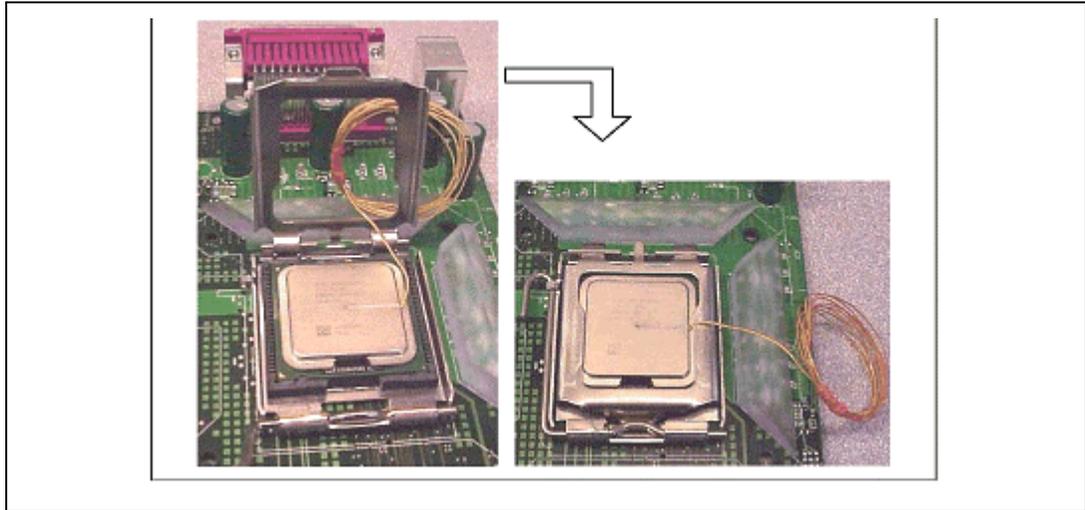


When installing the processor into the socket, make sure that the thermocouple wires exit above the load plate. Pinching the thermocouple wires between the load plate and the IHS will likely damage the wires.

Note: When thermocouple wires are damaged, the resulting reading may likely be wrong. For example, if there are any cuts into the wires insulation where the wires are pinched between the IHS and the load plate, the thermocouple wires can get in contact at this location. In that case, the temperature would be really measured will be measured on the edge of the IHS/socket load plate area. This temperature will likely be much lower than the temperature at the center of the IHS.

Prior to installing the heatsink, make sure that the thermocouple wires remain below the IHS top surface, by running a flat blade on top of the IHS for example.

Figure B-14. Thermocouple Wire Management



B.9 Local Air Thermocouple Placement

For passive heatsinks, two thermocouples will be placed 10 mm upstream of the processor heatsink. The thermocouples will be centered with respect to the height of the heatsink fins and evenly across the width of the heatsink.

For active heatsinks, four thermocouples will be placed on the fan inlet. These thermocouples will be mounted between 5 mm and 10 mm above the fan. The average of these measurements will be used to represent the local inlet temperature to the active heatsink.

Figure B-15. Local Air Thermocouple Placement for Passive Heatsinks

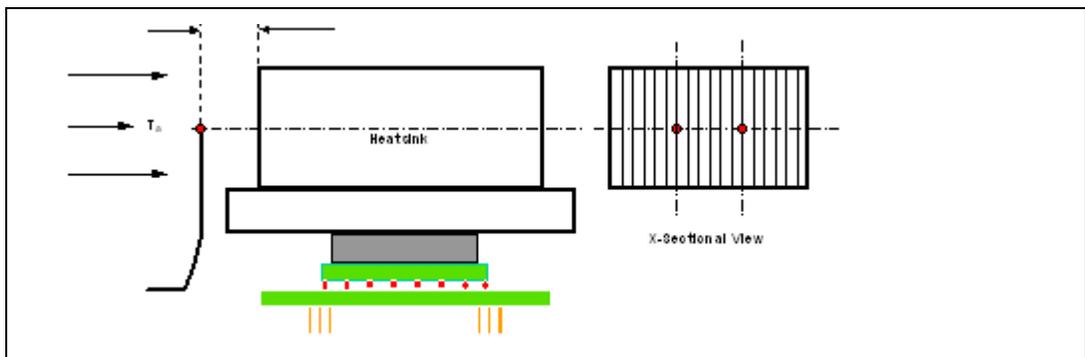


Figure B-16. Local Air Thermocouple Placement for Active Heatsinks (Side View)

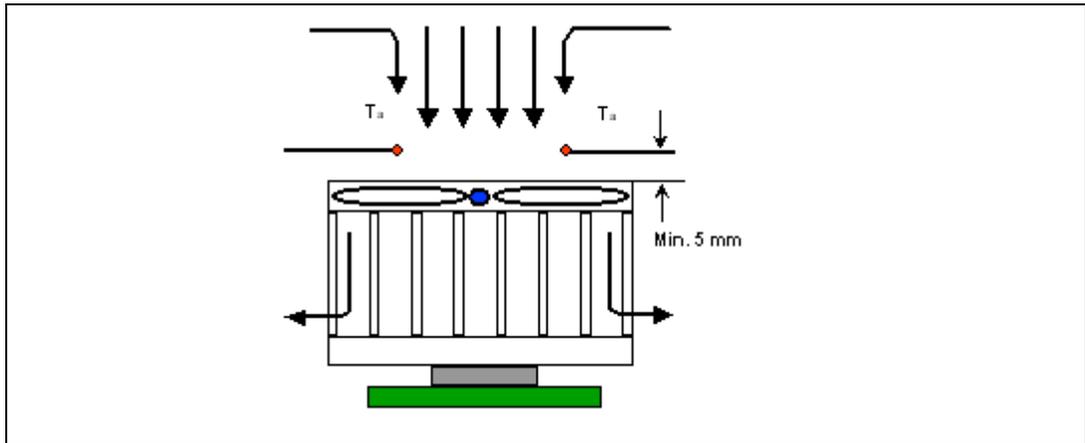
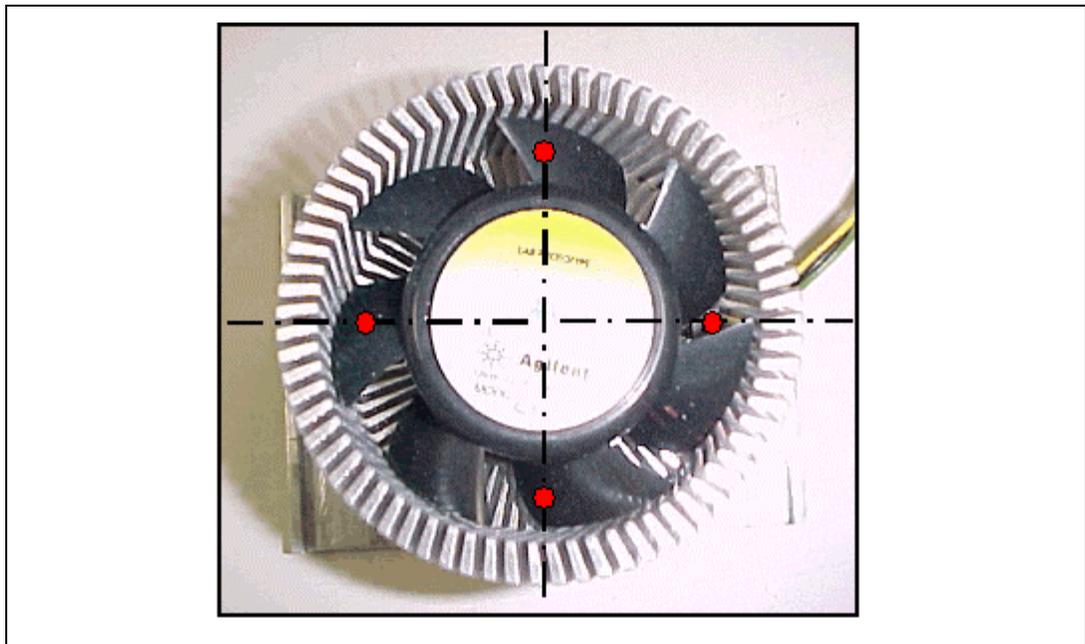


Figure B-17. Local Air Thermocouple Placement for Active Heatsinks (Plan View)





C Safety Requirements

Heatsink and attachment assemblies shall be consistent with the manufacture of units that meet the safety standards:

- UL Recognition-approved for flammability at the system level. All mechanical and thermal enabling components must be a minimum UL94V-2 approved.
- CSA Certification. All mechanical and thermal enabling components must have CSA certification.
- Heatsink fins must meet the test requirements of UL1439 for sharp edges.

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D Quality and Reliability Requirements

D.1 Intel Verification Criteria for the Reference Designs

D.1.1 Reference Heatsink Thermal Verification

The Intel reference heatsinks will be verified within specific boundary conditions based on the methodology described in Appendix B.

The test results, for a number of samples, are reported in terms of a worst-case mean + 3σ value for thermal characterization parameter using real processors.

D.1.2 Environmental Reliability Testing

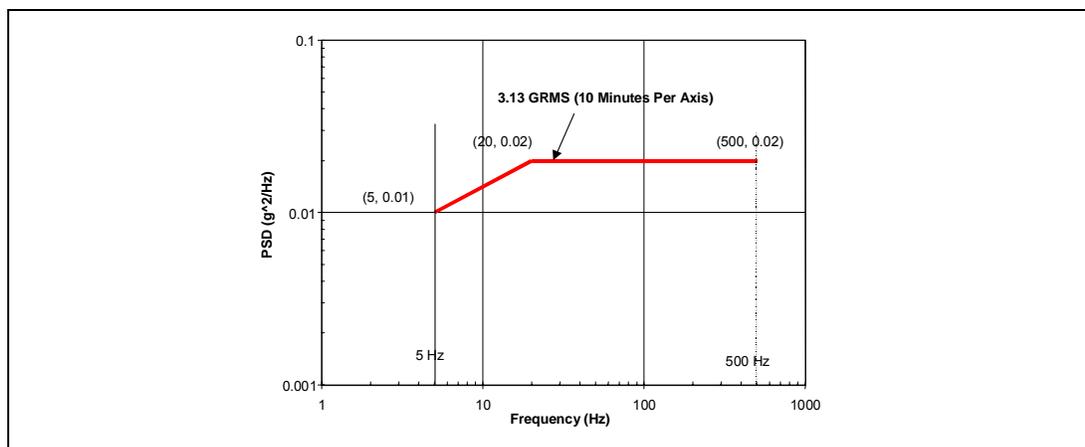
D.1.2.1 Structural Reliability Testing

Structural reliability tests consist of unpackaged, board-level vibration and shock tests of a given thermal solution in assembled state, as well as long-term reliability testing (temperature cycling, bake test). The thermal solution should be capable of sustaining thermal performance after these tests are conducted; however, the conditions of the tests outlined here may differ from the customers' system requirements.

D.1.2.2 Random Vibration Test Procedure

- Duration: 10 min/axis, 3 axes
- Frequency Range: 5 Hz to 500 Hz
- Power Spectral Density (PSD) Profile: 3.13 G RMS (refer to Figure D-1).

Figure D-1. Random Vibration PSD

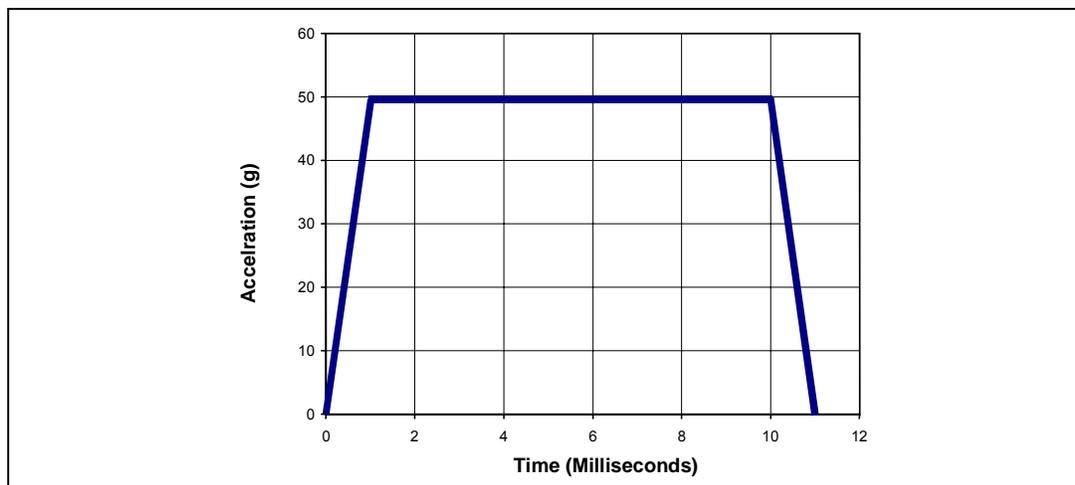


D.1.2.3 Shock Test Procedure

Recommended performance requirement for a baseboard:

- Quantity: 3 drops for + and – directions in each of 3 perpendicular axes (i.e. total 18 drops).
- Profile: 50 G trapezoidal waveform, 11 ms duration, 4.32 m/sec minimum velocity change.
- Setup: Mount sample board on test fixture.

Figure D-2. Shock Acceleration Curve



D.1.2.4 Recommended Test Sequence

Each test sequence should start with components (i.e. baseboard, heatsink assembly, etc.) that have not been previously submitted to any reliability testing.

The test sequence should always start with a visual inspection after assembly, and BIOS/Processor/memory test. The stress test should be then followed by a visual inspection and then BIOS/Processor/memory test.

D.1.2.5 Post-Test Pass Criteria

The post-test pass criteria are:

1. No significant physical damage to the heatsink and retention hardware.
2. Heatsink remains seated and it's bottom remains mated flatly against the IHS surface. No visible gap between the heatsink base and processor IHS. No visible tilt of the heatsink with respect to the retention hardware.
3. No signs of physical damage on baseboard surface due to impact of heatsink.
4. No visible physical damage to the processor package.
5. Successful BIOS/Processor/memory test of post-test samples.
6. Thermal compliance testing to demonstrate that the case temperature specification can be met.

D.1.2.6 Recommended BIOS/Processor/Memory Test Procedures

This test is to ensure proper operation of the product before and after environmental stresses, with the thermal mechanical enabling components assembled. The test shall be conducted on a fully operational baseboard that has not been exposed to any battery of tests prior to the test being considered.

Testing setup should include the following components, properly assembled and/or connected:

- Appropriate system baseboard.
- Processor and memory.
- All enabling components, including socket and thermal solution parts.

The pass criterion is that the system under test shall successfully complete the checking of BIOS, basic processor functions and memory, without any errors. *Intel PC Diags* is an example of software that can be utilized for this test.

D.1.3 Material and Recycling Requirements

Material shall be resistant to fungal growth. Examples of non-resistant materials include cellulose materials, animal and vegetable based adhesives, grease, oils, and many hydrocarbons. Synthetic materials such as PVC formulations, certain polyurethane compositions (e.g. polyester and some polyethers), plastics which contain organic fillers of laminating materials, paints, and varnishes also are susceptible to fungal growth. If materials are not fungal growth resistant, then MIL-STD-810E, Method 508.4 must be performed to determine material performance.

Material used shall not have deformation or degradation in a temperature life test.

Any plastic component exceeding 25 grams must be recyclable per the European Blue Angel recycling standards.

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E Enabled Suppliers Information

Component	Description	Development Suppliers	Supplier Contact Info
Cooling Solution Heatsink	Copper Fin, Copper Base	Fujikura* (stacked fin) CNDA 36187 Furukawa (crimped fin) CNDA 65755	Mechatronics* Steve Carlson 800-453-4569 x205 steve@mechatronics.com Furukawa America Katsu Mizushima (408) 232-9306 katsumizushima@mindspring.com
Thermal Interface Material	Grease	Shin-Etsu* G751 CNDA 75610	Donna Hartigan (480) 893-8898
Cooling Solution Spring	Stainless Steel 301, Kapton* Tape on Spring Fingers	ITW Fastex* CNDA 78538 AVC* CNDA 2085011 Foxconn* CNDA 11251	Ron Schmidt (847) 299-2222 rschmidt@itwfastex.com Felicia Lee 886-2-22996390 x144 felicia@avc.com.tw

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